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## **Test Setup for Axially Loaded Piles in Sand**

Thomassen, Kristina

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# Test Setup for Axially Loaded Piles in Sand

## **Kristina Thomassen**



## Aalborg University Department of Civil Engineering Division of Structures, Materials and Geotechnics

**DCE Technical Report No. 195** 

## **Test Setup for Axially Loaded Piles in Sand**

by

Kristina Thomassen

October 2015

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## Contents

1	Ob	jective	4
2	Te	st Setup for axially loaded piles in sand	4
	2.1	Sand Box	5
	2.2	Soil Properties	6
	2.3	Pile Specifications	7
	2.4	Increasing Effective Stresses	7
3	Sp	ecifications of Test Parts	9
4	Da	ta Acquisition Devices	9
	4.1	HBM Spider8	9
	4.2	HBM MGCplus	10
	4.3	MOOG Modular Test Controller	11
5	Da	ta Acquisition Software	11
6	De	vices for Soil Preparation	12
	6.1	Ascension Pipe	12
	6.2	Rod Vibrator	12
	6.3	CPT-device	12
7	Ну	draulic Systems	13
	7.1	Hydraulic System for Installation, Uninstallation and CPT	13
	7.2	Hydraulic System for Tests	14
8	Me	easuring Devices and Configuration for Installation	15
	8.1	Displacement Transducer WS15KT	16
	8.2	Load Cell 1	17
9	Me	easuring Devices and Configuration for CPT	19
	9.1	CPT Device	19
1(	)	Measuring Devices and Configuration for Tests	20
	10.1	Load Cell 2	22
	10.2	Displacement of Hydraulic Cylinder 2	22
	10.3	Displacement Transducers WS10-1 and WS10-2	23
	10.4	Pore Pressure Transducers	24
	10.5	Membrane Suction Transducer	25
	10.6	Air Pressure Transducer	26

## Test Setup for Axially Loaded Piles in Sand

10.7	7 Vacuum Pump	27
10.8	Pressurised Tube	27
11	Measuring Devices and Configuration for Uninstallation	27
12	Miscellaneous Equipment	28
12.1	L Aluminium Frames	28
12.2	Por workspace	28
12.3	3 Transition Pieces	28
12.4	Felt Cloth and Membrane	29
12.5	5 Tools	29
13	Bibliography	30

Test Setup for Axially Loaded Piles in Sand

## 1 Objective

The test setup for testing axially static and cyclic loaded piles in sand is described in the following. The purpose for the tests is to examine the tensile capacity of axially loaded piles in dense fully saturated sand. The pile dimensions are chosen to resemble full scale dimension of piles used in offshore pile foundations today.

## 2 Test Setup for axially loaded piles in sand

Figure 1 shows the test setup and Figure 2 a principal sketch of the test setup.



Figure 1: Test setup with applied suction.

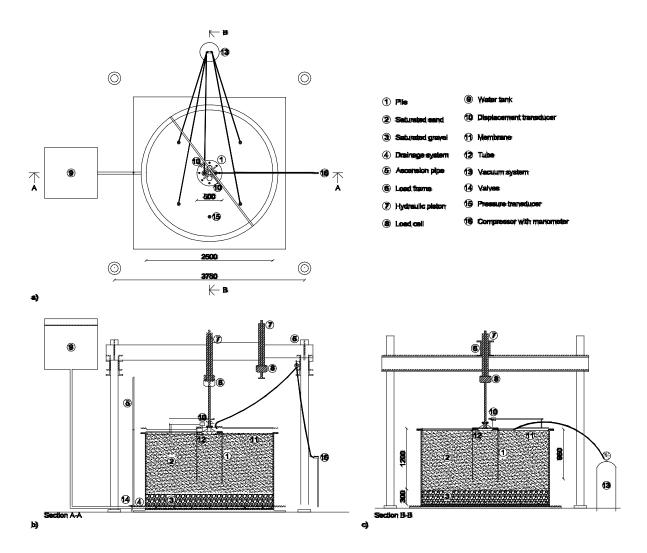


Figure 2: a) Layout of the test setup; b) Section A–A cross-sectional view of the test setup; c) Section B–B cross-sectional view of the test setup.

#### 2.1 Sand Box

The sand box has internal dimensions of D = 2.5 m and h = 1.2 m. To get saturated sand, it is necessary to let water in and out of the sand box, therefore, a drainage system is placed in the bottom of the sand box. The drainage system consists of perforated pipes equally placed in 30 cm of gravel. The gravel is covered by a felt cloth to prevent the sand to enter the drainage system. Above the felt cloth, 1.2 m of Baskarp Sand No. 15 is placed.

The pipes in the drainage system are connected to a water tank located above the sand box. From this tank, water is led into the sand box while controlling the gradient of the water through the sand by monitoring the water head in the ascension pipe, cf. Figure 3.



Figure 3: The drainage pipes in the bottom of the sand box. The water tank. The ascension pipe.

## 2.2 Soil Properties

When doing laboratory testing of offshore foundations at Aalborg University, Baskarp Sand No. 15 is used. The material properties for this type of sand are well-defined from previous tests at Aalborg University (Borup og Hedegaard 1995). However, for the presented test setup, a new load of Baskarp Sand No. 15 was purchased in 2012. The new sand is also Baskarp Sand No. 15 and Figure 4 shows that the sieve analyses give the same grain distribution as the sand from 1995. However, no classification tests are conducted on the new load of sand, and it is just assumed that the two sands have the same properties.

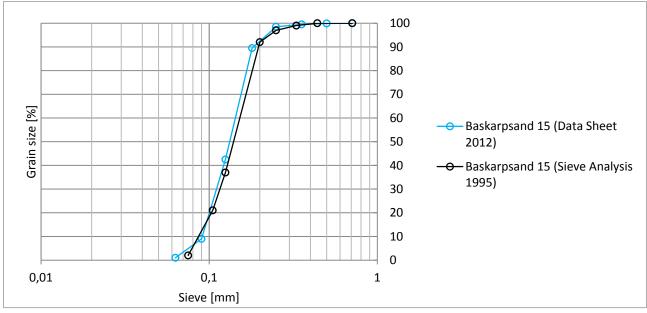


Figure 4: Sieve analyses for the old and new Baskarp Sand No. 15 (Borup og Hedegaard 1995) (SibelcoNordic 2008).

Table 1: Material properties for Baskarp Sand No. 15 (Borup og Hedegaard 1995).

Specific grain density	$d_{\scriptscriptstyle S}$	$[g/cm^3]$	2.64
Maximum void ratio	$e_{max}$	[-]	0.854
Minimum void ratio	$e_{min}$	[-]	0.549
50%-quantile	$d_{50}$	[mm]	0.14
Uniformity coefficient	$c_u = d_{60}/d_{10}$	[-]	1.78

#### 2.3 Pile Specifications

Figure 5 shows the pile segment is made of steel and is 1 m long and has a diameter of 0.5 m. The wall thickness is 3 mm, which gives a diameter to wall thickness ratio of 17. According to (Randolph og Gourvenec 2011) the smallest offshore piles have a diameter from around 0.76 m and beyond and diameter to wall thickness ratios vary between 25 and 100. Therefore, the pile segment has a diameter which is smaller than the piles used in reality but is much closer to full scale than piles usually used in laboratory tests to understand the processes under axial loading. The smaller diameter to wall thickness ratio of the specimen compared to full scale is not important as long as it is large enough to avoid instability of the pile wall during use. Besides the ratio only has an impact on the base resistance of the pile and the test pile is loaded in tension only.

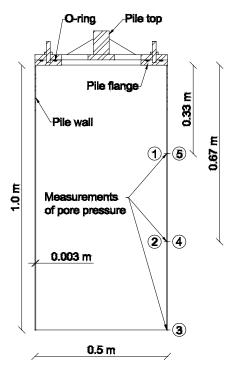


Figure 5: Principal sketch of the pile segment.

The pore pressure along the pile shaft is measured 1/3 L and 2/3 L from the pile top both on the inside and on the outside of the pile shaft. Moreover it is measured at the tip of the pile. Figure 5shows the positions of the pipes 1-5 connected to the pore pressure transducers on the pile. The pore pressure transducers placed on top of the pile. The specifications of the pore pressure transducers are given in Section Figure 5.

Figure 5 shows the position of pore pressure measurements. The external displacement transducers WS10-1 and WS10-2 are during tests positioned opposite each other connected to the nuts tightening the pile lid and the pile flange.

#### 2.4 Increasing Effective Stresses

The effective stresses in the soil are increased by covering the soil surface inside and outside the pile segment with a rubber membrane and applying suction underneath it. Hereby, overlaying soil layers are simulated due to the increase of effective stresses. Thereby, the soil-pile interface behaviour can be

investigated at various soil depths. To make sure that the suction is applied equally over the sand layer and to avoid that sand is sucked into the tubes, a felt cloth is placed between the sand and the membrane.

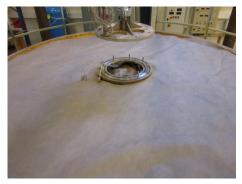


Figure 6: Felt cloth positioned on the sand surface.

The suction is controlled by a vacuum pump. The manometer on the vacuum system can be used as an indicator when regulating the pressure. However, to get the precise value of the suction it is necessary to attach a pressure transducer to the membrane and connect it to the MGCplus and monitor the pressure in Catman.



Figure 7: Vacuum system.

The suction is applied through four hoses connected to the membrane by means of quick-couplings, cf. Figure 8. The membrane is tightened at the edge of the sandbox by an aluminium frame, in the edge is a groove and the membrane is pressed into this groove between two O-rings making the connection air tight, cf. Figure 9a. The same assembly technique is used to tighten the membrane to the pile leaving room for holes in the membrane to lead the pore pressure pipes and bolts used to assemble the pile to the loading plate through, cf. Figure 9b. Here it is necessary with two O-rings, one on each side of the holes. It is important that the sand is 100% saturated and that very few air bobbles are present in the sand before applying the suction because the air expands under vacuum and will press the water out of the sand through the suction hoses. This will result in partially drained instead of fully saturated sand.



Figure 8: Membrane on sand surface connected to the vacuum pump by hoses.

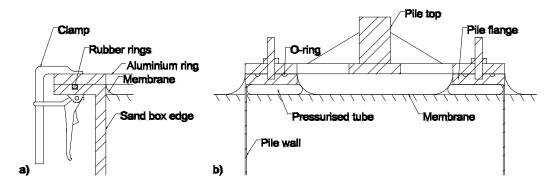


Figure 9: Sealing of the membrane at a) the sand box edge and b) at the pile flange.

## 3 Specifications of Test Parts

The test procedure consists of five parts:

- Installation
- Soil preparation
- CPT
- Test
- Uninstallation

The necessary equipment and specifications are given in the tables in Appendix A.

## 4 Data Acquisition Devices

## 4.1 HBM Spider8

Spider8 is used for data acquisition from CPT device, Load cell 1 and WS17kt.

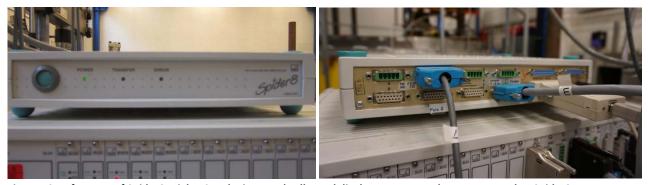


Figure 10: Left: Front of Spider8. Right: CPT device, Load cell 1 and displacement transducer connected to Spider8



Figure 11: The CPT device is connected to Spider8 through the box on the load frame through the socket "Cone" and the displacement transducer is connected via the socket "WS17kt".

## 4.2 HBM MGCplus

The MGCplus is used for data acquisition from the pore pressure transducers, the air pressure transducer, the membrane pressure transducer, the real pile head displacement and the load and displacement recorded by MOOG.



Figure 12: Left: Front of MGCplus. Right: the pore pressure transducers, the air pressure transducer, the membrane pressure transducer, the real pile head displacement and the load and displacement recorded by MOOG connected to MGCplus.



Figure 13: The displacement transducers are connected to MGCplus via the sockets "WS10-1" and "WS10-2". The pore pressure transducers are connected through CH6-1, CH6-2, CH6-5, CH6-6 and CH6-7. The membrane pressure transducer is connected to MGCplus via CH6-8.

#### 4.3 MOOG Modular Test Controller

The modular test controller is connected to the MOOG computer, load cell 1, the displacement transducer and the MGCplus. The test specifications is given in the program MOOG and executed by the modular test controller.



Figure 14: MOOG Modular Test Controller.

## 5 Data Acquisition Software

The measurements of the devices connected to Spider8 and MGCplus are recorded in Catman 6.0.

The test sequences are made and measurements of the devices connected to the Modular Test Controller are recorded in the MOOG Integrated Test Suite V.2.6.4.

## 6 Devices for Soil Preparation

## 6.1 Ascension Pipe





#### 6.2 Rod Vibrator

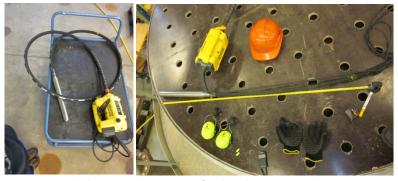


Figure 15: Rod vibrator and equipment for the vibration procedure.

#### 6.3 CPT-device

The dimensions of the CPT are given in Figure 12. The cone has a diameter of 15 mm and a 30 decrease inclination. The CPT device can only measure the cone resistance and not the sleeve friction and the pore pressure. The cone resistance is measured by four strain gauges placed in a full bridge of the type TML FLE-1-11 with gauge factor 2.03, cf. Figure 13. A more detailed description of the CPT—device and the basis for interpretation of soil parameters for Baskarp Sand No. 15 (aka Aalborg Universitets Sand No 1.) from the CPT measurements is found in Larsen (2008, App. A).

The CPT must be re-calibrated regularly. The calibration procedure is described in App. A. Interpretation of the CPT measurements based on (Larsen 2008) and (Ibsen, et al. 2009) is given in App. B.



Figure 16: CPT-device.

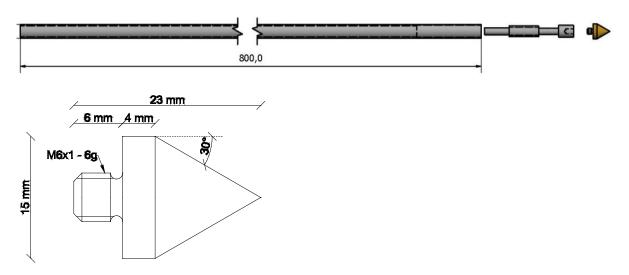


Figure 17: Dimensions of the CPT bar and cone.



Figure 18: Strain gauges and wiring inside the CPT to measure the cone resistance.

## 7 Hydraulic Systems

## 7.1 Hydraulic System for Installation, Uninstallation and CPT

Hydraulic cylinder 1, which is used during installation, uninstallation and CPTs, is the one to the right on Figure 1. The pump pressure is regulated according to the measured displacement rate of the cylinder.



Figure 19: Hydraulic station.

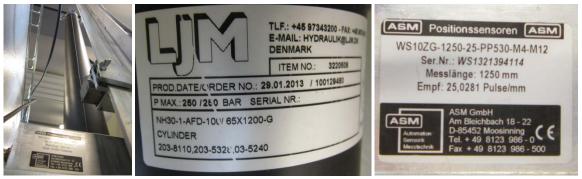


Figure 20: Hydraulic cylinder 1 and position transducer.

The cylinder can only be displacement controlled. The speed is adjusted on the black knob in the upper left corner of the switchboard. The intended speed is then displayed below the knob. While moving hydraulic cylinder 1 down the actual speed is measured and displayed to the right of the knob. The speed can then be adjusted during driving until an actual speed of e.g. 5 mm/s is reached. On the controller for hydraulic cylinder 1 two different speeds can be used. If the black knop on the controller is in horizontal or slightly upward position, the speed of hydraulic cylinder 1 corresponds to the speed chosen on switchboard. If the knob on the controller is turned downwards the hydraulic cylinder moves at maximum speed which is 6.18 mm/s.



Figure 21: Switchboard and controller for hydraulic cylinder 1.

## 7.2 Hydraulic System for Tests

Hydraulic cylinder 2 used for tests is the one to the left on Figure 1. The hydraulics is turned on at the switchboard. The pressure in the cylinder rises from 0 bars to 200 bars.



Figure 22: Hydraulic station and switchboard.

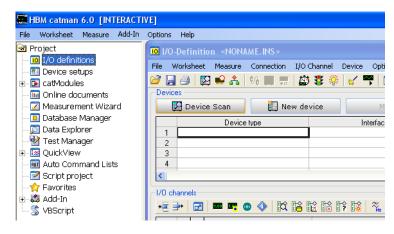
Hydraulic cylinder 1 can either be displacement or force controlled.

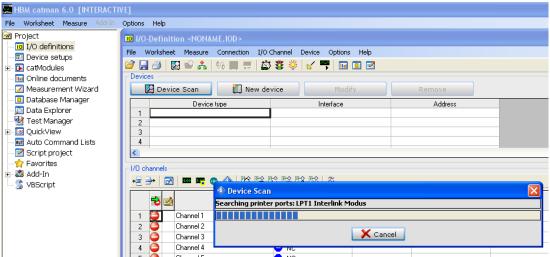


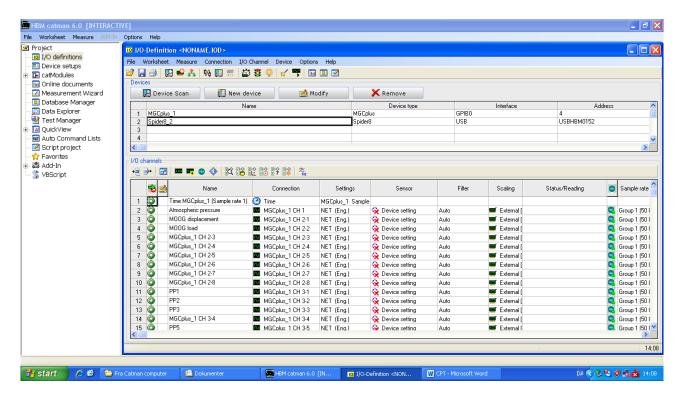
Figure 23: Hydraulic cylinder 2.

## 8 Measuring Devices and Configuration for Installation

In order to configure the different measuring devices in Catman it is necessary to make a device scan in Catman to locate Spider8 and MGCplus connected to the computer. Open Catman and press device scan.







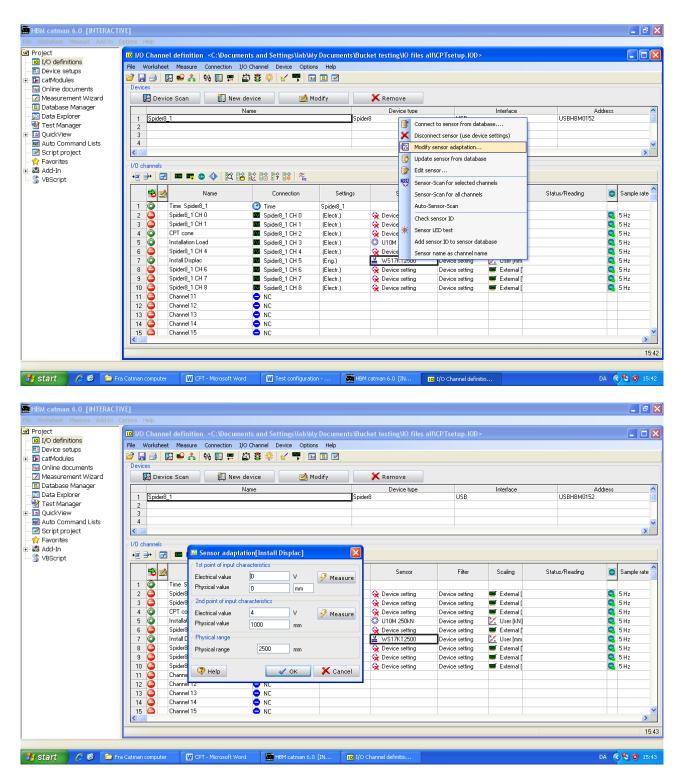
## 8.1 Displacement Transducer WS15KT

The displacement of hydraulic cylinder 1 is measured by a displacement transducer with a max range of 2500 mm. The transducer is connected to the Spider8 and the measurements are recorded in Catman.



Figure 24: Displacement transducer for hydraulic cylinder 1.

The configuration in Catman is as follows.



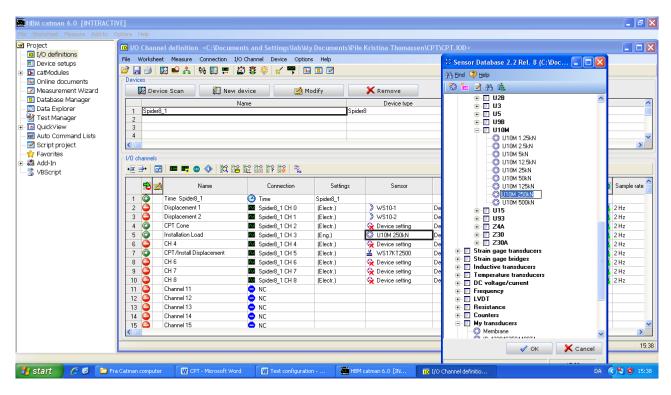
#### 8.2 Load Cell 1

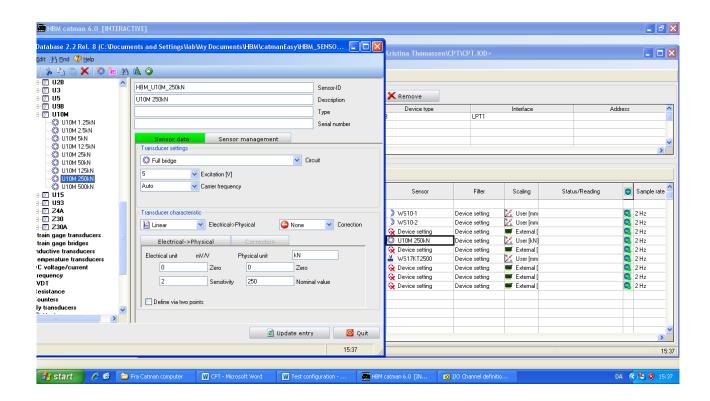
Load cell 1 is connected to Spider8 and the measurements are recorded by Catman.



Figure 25: Load cell 1.

The load cell is a standard HBM device and its configuration can be found in the sensor database within Catman as follows. Right click on the Sensor column and chose "Connect to sensor from database".



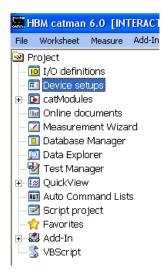


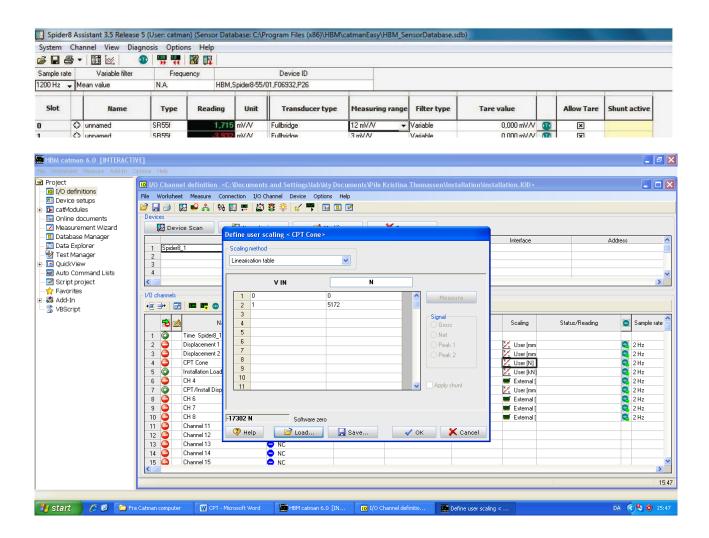
## 9 Measuring Devices and Configuration for CPT

The configuration of the displacement transducer WS15KT and Load cell 1 is presented in Chapter 8.

#### 9.1 CPT Device

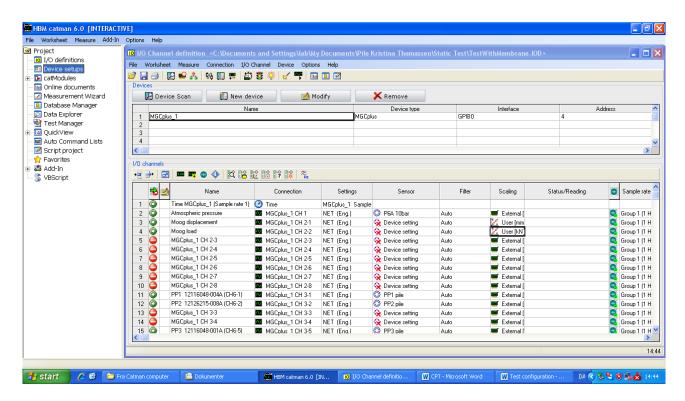
For the configuration of the CPT devise open the Device Setup and set the measuring range to 12 mV/V. Return to the I/O Definition and in Scaling choose User and define a linear relationship between the output in mV/V and Newton given by the calibration factor found by the procedure described in Appendix B.



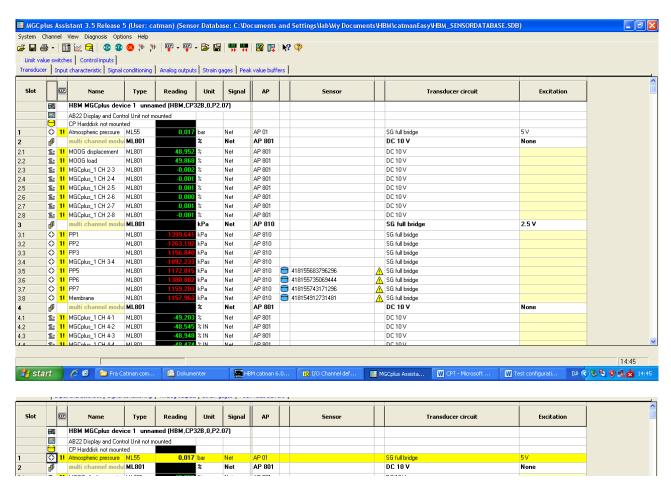


## 10 Measuring Devices and Configuration for Tests

The configuration of the measuring devices in Catman is started by turning on the MGCplus starting Catman and making a Device Scan. Both used and unused channels of the MGCplus connection boards are displayed in I/O definitions. The names of the channels of the connected measurement devices can be changed appropriately.



After the configuration of the measuring devises as described in the following, check the Device setups. Make sure that the Transducer circuit and the Excitation is correct for all devices.





#### 10.1 Load Cell 2

The load cell is connected to the Modular Test Controller and the measurements are recorded by both MOOG and Catman.



Figure 26: Load cell 2.

Like Load cell 1, Load cell 2 is a standard HBM transducer and can be found in the same manner in the Catman Database.

## 10.2 Displacement of Hydraulic Cylinder 2

The vertical position of hydraulic cylinder 2 is measured by a displacement transducer connected to the MOOG Modular Test Controller.



Figure 27: Displacement transducer for Hydraulic cylinder 2.

## 10.3 Displacement Transducers WS10-1 and WS10-2

The real displacement of the pile head is measured by means of two ASM WS10 displacement transducers with a range of 0–125 mm. The displacement transducers are connected to Spider8 and the data are recorded in Catman.



Figure 28: Left: Bar on which the transducers are positioned. Right: Position of the displacement transducers above the pile lid.

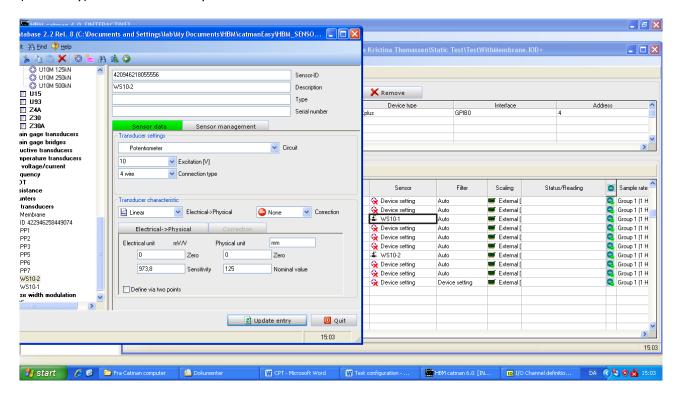


Figure 29: Displacement transducer WS10-1.



Figure 30: Displacement transducer WS10-2.

It is possible to define your own set of transducers in the HBM database. Right click on the Sensor column and chose "Connect to sensor from database" and press New. It is then possible to define your own transducer. The displacement transducers are defined as potentiometers and the calibration factor (sensitivity) is defined manually.



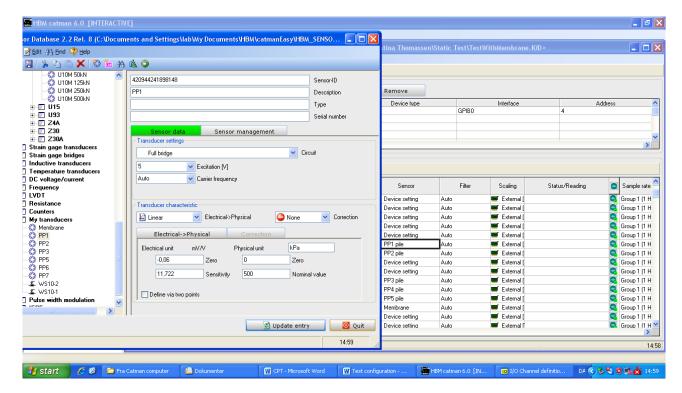
#### **10.4 Pore Pressure Transducers**

The calibration factors for each pore pressure transducer are given from the manufacturer for each transducer. Hence, it is important to change the calibration factor, if any of the transducers are replaced by another. The transducer is connected to the MGCplus and the data is recorded in Catman.



Figure 31: Example of pore pressure transducer.

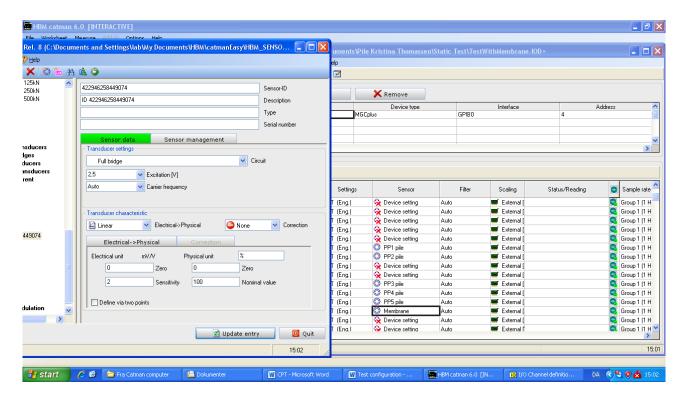
The pore pressure transducers can also be defined as a new device in the database.



## 10.5 Membrane Suction Transducer



The membrane suction transducers can also be defined as a new device in the database.



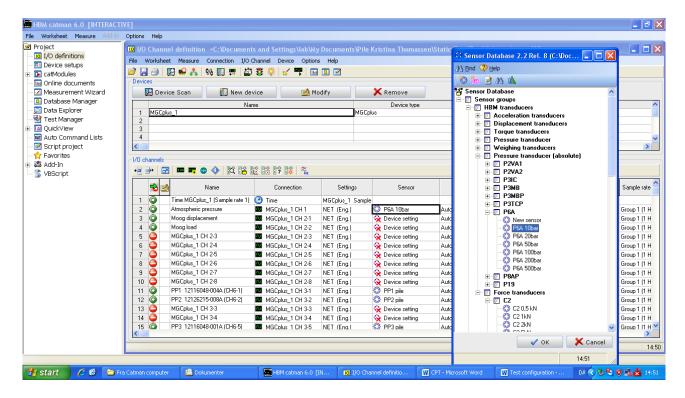
#### **10.6 Air Pressure Transducer**

To separate the changes in pore pressure from the changes in air pressure in the laboratory, an absolute pressure sensor of the type HBM p6a 10 bar 2mV/V is placed next to the test setup during tests. The transducer is connected to the MGCplus and the data is recorded in Catman.



Figure 32: Air pressure transducer.

The air pressure transducer is a standard HBM transducer and can be found in the database.



## 10.7 Vacuum Pump



#### 10.8 Pressurised Tube



## 11 Measuring Devices and Configuration for Uninstallation

See Chapter 8 about installation.

## 12 Miscellaneous Equipment

#### 12.1 Aluminium Frames



Figure 33: Aluminium frame 1 (to the left) is used during installation and uninstallation of the pile, sand preparation, and when conducting CPTs. Aluminium frame 2 (to the right) is used during tests.

#### 12.2 For workspace

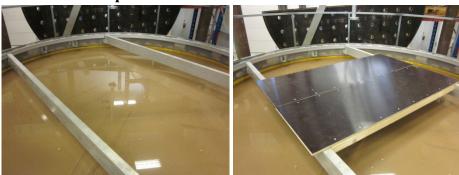


Figure 34: Aluminium bars no. 7 and 8 and footbridges.

## 12.3 Transition Pieces



Figure 35: Transition piece 1 (to the left) is used when conducting CPTs. Transition piece 2 (to the right) is used when installing or uninstalling the pile.



Figure 36: Transition piece for operating the pile with the ceiling crane.

## 12.4 Felt Cloth and Membrane



Figure 37: Felt cloth and Membrane

## **12.5 Tools**



Figure 38: Brush for cleaning the pile.



Figure 39: Screwdrivers etc. for moving the rigs with the hydraulic pistons.



Figure 40: Compressor for cleaning the pore pressure pipes and the sand box edge.



Figure 41: Clamp for tightening the aluminium frames to the sand box edge.



Figure 42: Ratchet for connecting the pile and the CPT to the transition pieces and for tightening the rig with Hydraulic piston 1 to the load beam when conducting CPTs.

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Thomassen, Kristina. *Test procedure for Axially Loaded Piles in Sand.* DCE Technical Report No. 196, Aalborg University, 2015.

#### **Equipment and Specifications** Appendix A

Table 2: Installation and uninstallation equipment and specifications.

Specifications	-	Unit		
Installation rate	6	mm/s		
Gradient	0,9i <sub>crit</sub>	-		
Measurements	Unit	Measured by	Connected to	
Displacement	mm	WS17KT	Spider8	
Load	kN	Load cell 1	Spider8	
Time	S			
Equipment	Manufacturer	Туре	Specification	Calibration factor
Hydraulic pump 1	Hydra	1SPA1 2-S	250 bar	
Hydraulic cylinder 1	IJМ		250 bar	
Position transducer	ASM	WS10ZG-1250-25- PP530-M4-M12	1250 mm	25.0291 pulse/mm
Displacement transducer	ASM	WS17KT-2500-10V- L10-M4-M12	2500 mm	4.000 V/m
Load cell 1	НВМ	U10M	250 kN	2.119 mV/V
Ceiling crane				
Pile	AAU			
Transition piece 2	AAU			
Aluminium ring 1	AAU			
Data acquisition	Manufacturer	Туре		
Spider8	НВМ	Spider8		
Catman	НВМ	Catman 6.0		

Specifications		Unit
Vibrator penetration rate	2	min/hole
Gradient	0,9	-
Equipment	Manufacturer	Туре
Rod vibrator	Wacher Neuson	IRFU 45 (rod vibrator) M2000 (motor)
Earmuffs		
Vibration gloves		
Stopwatch		
Helmet		

Table 4: CPT equipment and specifications.

Specifications		Unit		
Penetration rate	5	mm/s		
Measurements	Unit	Measured by	Connected to	
Cone resistance	N	CPT device	Spider8	
Displacement	mm	WS17kt	Spider8	
Load	kN	Load cell 1	Spider8	
Time	S			
Equipment	Manufacturer	Туре	Specification	Calibration factor
Hydraulic pump 1	Hydra	1SPA1 2-S	250 bar	
Hydraulic cylinder 1	IJМ		250 bar	
Position transducer	ASM	WS10ZG-1250-25- PP530-M4-M12	1250 mm	25.0291 pulse/mm
Displacement transducer	ASM	WS17KT-2500-10V- L10-M4-M12	2500 mm	4.000 V/m
Load cell 1	нвм	U10M	250 kN	2.119 mV/V
CPT device	AAU		4000 N	Must be calibrated on a regular basis, see App. A
Transition piece 1	AAU			
Data acquisition	Manufacturer	Туре		
Spider8	НВМ	Spider8		
Catman	НВМ	Catman 4.0		
Equipment CPT calibration	Manufacturer	Туре	Specification	Calibration factor
Load cell	НВМ	S9 5	5 kN	2mV/V
Lathe				

Table 5: Test equipment and specifications.

Size	Unit				
0,002	mm/s		=	-	
0,1	Hz				
1	Hz				
2	Hz				
Unit	Measu	red by	Connected to	Connection boards	Box name
S	=		-	-	_
S					
N	Load ce	ell 2	Modular Test controller MGCplus	AP801	
mm		•	Modular Test controller MGCplus	AP801	
mm			MGCplus	AP836i	WS10-1 WS10-2
kPa	PP1		MGCplus	AP810i	CH6-1
kPa	PP2		MGCplus	AP810i	CH6-2
			•		CH6-5
					CH6-6
			•		CH6-7
	_	rane	•		CH6-8
		iaiic			p6a
_			_	-	<del>-</del>
ivianutacturer	туре		Specification	factor	Zero poin
Hydra	1SPA1	2-S	250 bar	=	=
IJM			250 bar		
MooG	S106				
HBM	U10M		250 kN	2.1171 mV/V	0 mV
ASM	WS10-	125-R1K-L10-	125 mm	7.7879	0 mV
	SBO-D8	3		mV/V/mm	
ASM	WS10-125-R1K-L10-		125 mm	7.7904 mV/V/mm	0 mV
LISAB	NS-B (12126215-		5 bar	13.802 mV/V	0.270 mV
LISAB	NS-B	(12126215-	5 bar	13.702 mV/V	0.970 mV
LISAB	NS-B 006A)	(12126215-	5 bar	13.980 mV/V	-0.030 m\
LISAB	NS-B 013A)	(12126215-	5 bar	14.862 mV/V	-0.690 m\
LISAB	NS-B 001A)	(12116048-	5 bar	13.918 mV/V	0.300 mV
нвм	p6a		10 bar	2 mV/V/bar	
Manufacturer	Туре				
НВМ	MGCpl	us	-	-	-
MOOG					
HBM	Catmai	n 6.0			
MOOG					
	O,002 O,1 1 2 Unit  S S S N  mm  kPa kPa kPa kPa kPa kPa kPa kPa kPa kBa kPa kBa kBa kBa kBa kBa kBa kBa kBa kBa kB	0,002         mm/s           0,1         Hz           1         Hz           2         Hz           Unit         Measu           s         N           S         N           N         Load of           mm         Disp. of           cylinder         cylinder           mm         WS10-           kPa         PP1           kPa         PP2           kPa         PP3           kPa         PP4           kPa         PP5           kPa         Memb           kPa         PP4           kPa         PP5           kPa         Memb           kPa         PP4           kPa         PP5           kPa         Memb           kPa         PP5           kPa         Memb           kPa         PP4           kPa         PP5           kPa         Memb           kPa         NS-B           O04A)         LISAB           LISAB         NS-B           O01A)         NS-B           O01A)         NS-B	0,002       mmm/s         0,1       Hz         1       Hz         2       Hz         Unit       Measured by         S         S       S         Unit       Measured by         Measured by         Measured by         S         N         Load cell 2         mm         Disp. of hydraulic cylinder 2         Cylinder 2         Mm         WS10-1         WS10-1         WS10-2         RPA         WS10-2         WPA         WPA	0,002         mm/s           0,1         Hz           1         Hz           2         Hz           Unit         Measured by         Connected to           S         S           N         Load cell 2         Modular Test controller MGCplus           mm         Disp. of hydraulic Cylinder 2         MoGCplus           mm         WS10-1         MGCplus           wS10-2         MGCplus           kPa         PP1         MGCplus           kPa         PP2         MGCplus           kPa         PP3         MGCplus           kPa         PP4         MGCplus           kPa         PP5         MGCplus           kPa         Membrane         MGCplus           kPa         Membrane         MGCplus           kPa         Membrane         MGCplus           Manufacturer         Type         Specification           Hydra         1SPA1 2-S         250 bar           LIM         250 bar         125 mm           MooG         580-D8         125 mm           LISAB         NS-B (12126215- 5 bar           008A)         125 mm           LISAB	0,002

## Appendix B Calibration of CPT-device

The CPT device is calibrated before conducting a series of tests. The CPT device is fastened in a lathe. The cone is protected by a block of hard rubber with an indentation fitting the cone. A HBM force transducer type S9 5 of 5 kN = 2 mV/V (registration no. 54086) is inserted between the cone and the bench to measure the force on the cone when the lathe handle is turned.









The measurements of the force transducer and the CPT device, respectively, are recorded in Catman. The CPT output is in mV/V while the output from the force transducer is in N. The measuring range of the CPT should be 12 mV/V. Prior to the calibration test, the output of the CPT and the force transducer should be zeroed. Let Catman display a real-time graph and apply a force up to maximum 4 kN. Release the force and re-load a couple of times to validate the correlation between the force and the CPT output. Figure 43 shows a calibration example.

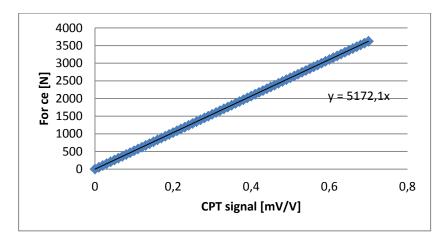


Figure 43: Example of determination of calibration factor.

## **Appendix C Interpretation of CPT results**

The CPT cone resistance,  $q_c$ , plotted against the depth is used to verify homogeneous compaction of the sand. This means that the cone resistances found from the five CPTs conducted before each test should be very similar. Moreover, the cone resistances should show similar soil conditions for all the conducted loading tests to ensure that the soil conditions and, thereby, the test results in the various tests are comparable.

From the CPT-results the following soil parameters are determined:

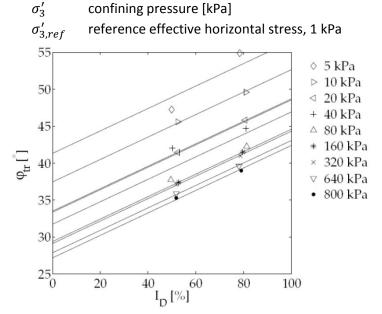
- Friction angle,  $\varphi_{tr}$  [°].
- Dilation angle,  $\psi_{tr}$  [°].
- Relative density,  $D_r$  [-].
- Effective unit weight,  $\gamma'$  [kN/m<sup>3</sup>].
- Initial stiffness of the sand,  $E_0$  [kPa].

The expressions for the internal angle of friction and the dilation angle are based on results from triaxial tests on Baskarp Sand No. 15 (Ibsen, et al. 2009). The friction angle and dilation angle are considered dependent on the density index and the confining pressure,  $\sigma'_3$ . The triaxial tests were performed with two different density indices and nine different confining pressures. The expressions for the friction angle and the dilation angle given below were determined by plotting the angles against the density index. The figure shows the friction angles plotted against the density index.

$$\varphi_{tr} = 15.2^{\circ} \cdot D_r + 27.39^{\circ} \cdot (\sigma_3'/\sigma_{3,ref}')^{-0.2807} + 23.21^{\circ}$$

$$\psi_{tr} = 19.5^{\circ} \cdot D_r + 14.86^{\circ} \cdot (\sigma_3'/\sigma_{3,ref}')^{-0.09764} - 9.946^{\circ}$$

Where:



Friction angle versus relative density (Ibsen, et al. 2009).

The density index is found by an iterative procedure involving the following four equations.

$$\gamma' = \frac{d_s - 1}{1 + e_{in-situ}} \gamma_w$$

$$\sigma_1' = \gamma' \cdot x$$

$$D_r = c_2 \left( \frac{\sigma_1' / \sigma_{1,ref}'}{\left( q_c / q_{c,ref} \right)^{c_1}} \right)^{c_3}$$

$$D_r = \frac{e_{max} - e_{in-situ}}{e_{max} - e_{min}}$$

Where:

 $\gamma'$ effective unit weight of soil [kN/m<sup>3</sup>] relative density [-],  $d_s = 2.64$ in-situ void ratio [-]  $e_{in-situ}$ unit weight of water [kN/m<sup>3</sup>]  $\gamma_w$ effective vertical stress [MPa]  $\sigma_1'$  $\sigma'_{1,\mathrm{ref}}$ reference effective vertical stress, 1 MPa cone resistance [MPa]  $q_c$ reference CPT cone resistance, 1 MPa  $q_{c,ref}$ depth below soil surface [m] constants [-],  $(c_1, c_2, c_3) = (0.75, 5.14, -0.42)$  $c_1,c_2,c_3$ maximum void ratio [-],  $e_{max} = 0.854$  $e_{max}$ minimum void ratio [-],  $e_{min} = 0.549$  $e_{min}$ 

As the in-situ void ratio and the effective unit weight are unknown, the relative density is found by inserting a guessed value of  $e_{in-situ}$  and iterate until the difference between two successive values of  $e_{in-situ}$  is less than  $10^{-4}$ .

The effective horizontal stress,  $\sigma'_3$ , is dependent on the effective vertical stress and the coefficient of horizontal earth pressure at rest,  $K_0$ .  $K_0$  depends on the friction angle, giving the following expression for  $\sigma'_3$ . Thereby, the friction angle is the only unknown in the expression for the friction angle and can then be found by iteration.

$$\sigma_3' = \sigma_1' \cdot K_0$$

$$= (\gamma' \cdot x + P_0) \cdot K_0$$

$$= (\gamma' \cdot x + P_0) \cdot (1 - \sin \varphi_{tr})$$

Where

 $K_0$  earth pressure coefficient at rest [-]  $P_0$  overburden pressure [kPa]

Inserting this expression into the formula for the friction angle shows that  $\varphi_{tr} \to \infty$  for  $\sigma_3' \to 0$ . Furthermore, the figure shows that the expression for the friction angle does not fit the triaxial test for  $\sigma_3' = 5$  kPa well. As the CPTs only reach a depth of 1 m,  $\sigma_3'$  lies between 0 kPa and 2 kPa for the tests without overburden pressure (i.e. without suction under the membrane). To be able to use the equation

for the friction angle,  $\sigma'_3$  is set to 5 kPa for the tests without overburden pressure. As a result, the friction angle may be judged slightly lower than the correct value, however, the difference is considered tolerable.

(Brinkgreve og Swolfs 2007) gives the following expression for the secant stiffness of the soil,  $E_{50}$ . This value of  $E_{50}$  is then used to find the tangential modulus,  $E_{0}$  (Ibsen, et al. 2009).

$$E_{50} = (0.6322 \cdot D_r^{2.507} + 10920) \left( \frac{c \cdot \cos \varphi_{tr} + \sigma_3' \cdot \sin \varphi_{tr}}{c \cdot \cos \varphi_{tr} + \sigma_3'^{ref} \cdot \sin \varphi_{tr}} \right)$$

$$E_0 = \frac{2 \cdot E_{50}}{2 - R_f}$$

Where:

 $E_{50}$  secant stiffness [kPa]

c cohesion [kPa]

 $\sigma_3^{\prime ref}$  reference horizontal stress [kPa],  $\sigma_3^{\prime ref}=100$ kPa

 $E_0$  tangential stiffness [kPa]

 $R_f$  ratio between  $q_f$  and  $q_a$  [-], the standard value is  $R_f=0.9$  cf. (Brinkgreve og Swolfs 2007)

 $q_f$  ultimate deviatoric stress [kPa]

 $q_a$  asymptotic value of the shear strength [kPa]

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