

Early Stages of Agglomeration and Deposition of Adhesive Micron-sized Particles in Fully-Developed Turbulent Pipe Flows

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Abstract

1 This study presents numerical results on how micron-sized adhesive particles
2 agglomerate and deposit in fully-developed turbulent pipe flows. Particle-
3 particle and particle-wall interactions are modelled using the mechanistically-
4 based soft-sphere Discrete Element Method (DEM) and fluid turbulence is
5 resolved using Large Eddy Simulations (LES). In this study, the adhesive
6 behaviour of particles, ultimately resulting in agglomeration and deposition of
7 particles, is predicted using JKR theory.

8 In this study, the agglomerating behaviour of mono-sized particles with con-
9 stant volume fraction $\phi = 0.001$ in turbulent flows with $\text{Re} = U \cdot D/\nu = 10,000$
10 is investigated. By varying the Stokes number $\text{St} = \rho_p d_p^2 U / (18\mu D)$ in the
11 range 0.4 to 25.6, the study presents results on how changes in dominant
12 collision mechanism affect the agglomeration rate. The results show highest
13 agglomeration rate for intermediate Stokes numbers where the accelerative-
14 correlated collision mechanism is dominant. At either extreme of lower or higher
15 Stokes number, the agglomeration rate is decreased. Furthermore, at low Stokes
16 numbers, the radial particle concentration is almost uniform throughout the flow
17 field. At higher Stokes numbers, particles tend to accumulate either accumulate
18 in the centre of the pipe or deposit on the wall.

19 Furthermore, to investigate the transition from weakly adhesive particles
20 to highly adhesive particles, the non-dimensional adhesiveness parameter
21 $\text{Ad} = \gamma / (\rho_p U^2 d_p)$ is varied at constant elasticity parameter $\lambda = E / (\rho_p U^2)$ and

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22 coefficient of restitution e . The results show a sharp increase in agglomeration
23 rate and fraction of particles captured by the wall.

Keywords: Adhesive particles, Agglomeration, Deposition, Turbulent pipe
flow, Large Eddy Simulation (LES), Discrete Element Method (DEM),
Johnson-Kendall-Roberts (JKR)

24 **1. Introduction**

25 The transport, agglomeration and subsequently deposition of small adhesive
26 particles play important roles in many industrial and fundamental processes.
27 These processes range from particles accumulating at heat transfer surfaces,
28 particles blocking pores in membrane filtration systems, particles being inhaled
29 and deposited in our lungs to interstellar medium agglomerating causing early
30 stages of new planets to form in space.

31 All agglomeration and deposition processes are a result of particles colliding
32 with one another or a wall. The mechanisms governing particle collisions of
33 non-adhesive particles in turbulent flows have been devoted much attention in
34 literature. These studies date back to Saffman and Turner [1] who investigated
35 the collision frequency in isotropic turbulent flows in the limiting case of finite
36 size tracer particles with response times much than fluid response time $\tau_p \ll \tau_f$
37 and Abrahamson [2] who investigated the collision frequency in the other
38 limiting case of heavy particles with $\tau_p \gg \tau_f$. For particles having $\tau_p \approx \tau_f$,
39 various correlations have been proposed in literature [3]. However, as particles
40 begin to adhere and the turbulent flow of interest is anisotropic, e.g. a pipe
41 flow, a common approach is to resolve only the turbulence scales that affect
42 motion of the particles considered using LES.

43 Different approaches exist to model the agglomeration process. One
44 approach is to represent agglomerates by equivalent spheres that grow in
45 size as the number of particles contained in the agglomerate increase [4, 5].
46 However, as shown by Brasil et al. [6], the morphology of agglomerates differs
47 significantly depending on how the agglomerates are formed, the properties

48 of the primary particles and properties of the fluid flow surrounding the
49 particles. Based on the Euler-Lagrangian approach, Sommerfeld and Stübing
50 [7] proposed a computational efficient agglomerate structure model. Using
51 this model, agglomerates are treated as point particles that carry additional
52 information such as locations of the primary particles and binding forces holding
53 the agglomerates together. Based on these properties, parameters such as
54 the convex hull and fractal dimension can be calculated and used to relate
55 agglomerate structure to flow resistance coefficients.

56 Another approach is to track each particle but assume the particles to stay
57 adhered when agglomerated or deposited on a wall. However, the DEM study
58 of laminar channel flow by Marshall [8] shows that phenomena such as bending
59 and break-off of agglomerates occur frequently and play important roles to
60 accurately predict the state where the rate of particles being re-entrained back
61 into the fluid asymptotically approaches the rate of particles being deposited.
62 Furthermore, the study suggests the mechanism of agglomerates continuously
63 breaking up as agglomerates are formed to be controlled by impacting particles
64 or agglomerates rather than fluid forces.

65 As noted in most studies coupling DEM to a fluid phase, there is typically a
66 large difference between the time step sizes required to resolve particle collisions
67 δt_{DEM} and fluid flows δt_{f} , so that $\delta t_{\text{col}} \ll \delta t_{\text{f}}$. As $\delta t_{\text{col}} = \mathcal{O}(10^{-9} \text{ s})$, this is
68 also valid for a wide range of turbulent flows. This fact is commonly used to
69 speed up coupled simulations by introducing softer particles by lowering the
70 particle stiffness and thereby making particle collisions take place over longer
71 time periods. However, as noted in studies by Kobayashi et al. [9], Gu et al.
72 [10], Hærvig et al. [11], depending on the adhesiveness model, introducing softer
73 particles should be accompanied by a lower adhesive forces in order for the
74 collision outcome (stick/rebound) to stay the same.

75 When the agglomerates increase in size, the study by Dizaji and Marshall
76 [12] shows that the local fluid velocity in an agglomerate becomes increasingly
77 correlated with the agglomerate velocity. In this case, a two-way coupling
78 between particles and fluid is needed to accurately represent the presence

79 of particles on the fluid. Furthermore, due to differences in agglomerate
80 morphology, it is not trivial to model the particle-fluid interaction without
81 resolving the flow fluid around each particle. Attempts to correlate particle
82 drag with particle volume fraction and Reynolds number include Ergun and
83 Orning [13] who experimentally correlated pressure gradients in fluidized beds
84 to the particle void fraction. Later studies by Hill et al. [14, 15] rely on the
85 Lattice-Boltzmann Method (LBM) to resolve the flow around particles and
86 correlate the drag force exerted on particles with particle volume fraction and
87 the Reynolds number. While this method is highly accurate for homogeneous
88 packing, the spatial variations in agglomerates that range from compact to
89 dendritic in structure complicate the formulation of a general drag model.
90 Dietzel and Sommerfeld [16] resolved the flow in agglomerates by local grid-
91 refined Lattice-Boltzmann Method (LBM) simulations and correlated the overall
92 drag force on different agglomerate morphologies to the projected cross section
93 of the convex hull perpendicular to the mean flow direction. However, as the
94 agglomerating and break-up mechanisms are governed by the particles being
95 affected by different fluid forces, this approach is not suitable for this study.

96 While numerous studies on two-way coupled particle-fluid interactions
97 have been reported, only a few account for the adhesive behaviour by fully
98 resolving collision using the soft-sphere DEM approach. Afkhami et al. [17]
99 studied the effect of particle adhesiveness using three different particle surface
100 energy densities and showed a direct link between surface energy density and
101 agglomeration rate.

102 The purpose of this study is look into how particle response time and particle
103 adhesiveness affect the agglomeration rate. To obtain a fully-developed flow, the
104 computational domain is made periodic in the stream-wise direction. To avoid
105 imposing any limitations on the agglomeration process, the soft-sphere DEM
106 approach is used to resolve how particles collide, agglomerate, deposit and are
107 re-entrained into the fluid due to collisions with other particles or by fluid forces.
108 Figure 1 gives an overview of these different processes.

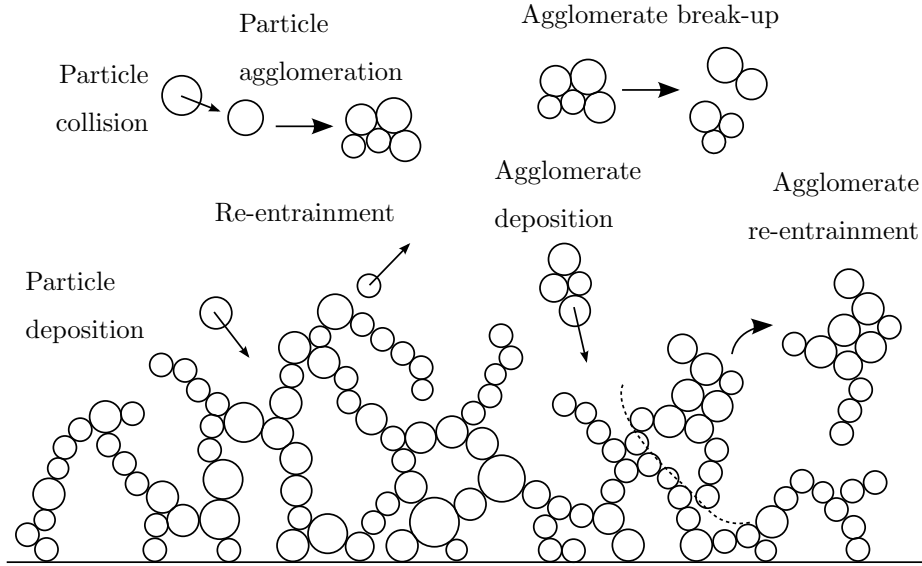


Figure 1: Overview of the different processes in the vicinity of a surface and away from a surface that are resolved directly by the soft-sphere DEM approach. The early stages of interest in this study are typically limited to particles colliding, particles agglomerating, agglomerates breaking up and a single particles depositing at surfaces.

109 2. Details on the numerical setup

110 2.1. Governing equations for fluid flow

111 The filtered LES equations governing unsteady, in-compressible, three-
 112 dimensional viscous fluid flow in a periodic domain with two-way coupling
 113 between particle and fluid phases are the continuity and momentum equations
 114 given by (1) and (2) respectively:

$$\frac{\partial(\alpha_f \bar{u}_i)}{\partial x_i} = 0 \quad (1)$$

$$\frac{\partial(\alpha_f \bar{u}_i)}{\partial t} + \frac{\partial(\alpha_f \bar{u}_i \bar{u}_j)}{\partial x_j} = -\frac{\alpha_f}{\rho_f} \frac{\partial \bar{p}}{\partial x_i} + \alpha_f \frac{\partial}{\partial x_j} \left((\nu + \nu_{\text{sgs}}) \frac{\partial \bar{u}_i}{\partial x_j} \right) + \beta \delta_{1,i} + f \quad (2)$$

115 where α_f denoting the local fluid volume fraction and $\beta \delta_{1,i}$ defines a momentum
 116 source term dynamically being changed to balance out the pressure gradient
 117 across the periodic domain. The sub-grid scale viscosity ν_{sgs} accounts for sub-
 118 grid scale turbulence and naturally approaches zero in the case of DNS. In this

119 study, focus is on LES where the sub-grid scale viscosity is modelled through
 120 the the wall-adapting local eddy-viscosity (WALE) model by Nicoud and Ducros
 121 [18]:

$$\nu_{\text{sgs}} = \left(C_w V^{1/3}\right)^2 \frac{\left(\bar{s}_{ij}^d \bar{s}_{ij}^d\right)^{3/2}}{\left(\bar{s}_{ij}^d \bar{s}_{ij}^d\right)^{5/2} + \left(\bar{s}_{ij}^d \bar{s}_{ij}^d\right)^{5/4}}, \quad (3)$$

122 where:

$$s_{ij}^d = \frac{1}{2} (\bar{g}_{ij}^2 + \bar{g}_{ji}^2) - \frac{1}{3} \delta_{ij} \bar{g}_{kk}^2, \quad \bar{s}_{ij} = \frac{1}{2} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right), \quad (4)$$

$$g_{ij} = \frac{\partial \bar{u}_i}{\partial x_j}, \quad g_{ij}^2 = g_{ik} g_{kj} \quad (5)$$

123 with constant $C_w = 0.325$ and where V is the local cell volume. Compared to
 124 the Smagorinsky-type models, the WALE model is more suited for wall-bounded
 125 flows, as the local sub-grid scale eddy-viscosity automatically goes to zero at the
 126 wall. Therefore, no dynamic constant adjusting or damping functions are needed
 127 to correct for walls.

128 2.2. Governing equations for particles

129 To model collisions of particles, the soft-sphere discrete element method
 130 (DEM) first proposed by Cundall and Strack [19] is used to track the
 131 translational and angular velocity throughout the simulations. For the i 'th
 132 particle with mass m_i , radius r_i and mass moment of inertia $I_i = (2/5)m_i r_i^2$,
 133 the position \mathbf{x}_i and angular velocity $\boldsymbol{\omega}_i$ are governed by:

$$m_i \frac{d^2 \mathbf{x}_i}{dt^2} = \mathbf{F}_{i,\text{con}} + \mathbf{F}_{i,\text{fluid}} \quad (6)$$

$$I_i \frac{d\boldsymbol{\omega}_i}{dt} = \mathbf{M}_{i,\text{con}} + \mathbf{M}_{i,\text{roll}} + \mathbf{M}_{i,\text{fluid}} \quad (7)$$

134 where the \mathbf{F}_{con} is a contact force upon collision and $\mathbf{F}_{\text{fluid}}$ is the combined fluid
 135 force acting on the particle. In the angular momentum equation, \mathbf{M}_{con} is the
 136 contact torque by a tangential off-set between colliding particles, \mathbf{M}_{roll} is the
 137 adhesive rolling resistance torque and $\mathbf{M}_{\text{fluid}}$ is the torque caused by interaction
 138 with the fluid phase.

139 *2.2.1. Contact forces*

140 Due to the small size of the particles of interest, the van der Waals force
 141 plays a dominant role in the collision process, which ultimately results in
 142 particles agglomerating and sticking to walls. Based on properties and size
 143 of the particles, particle deformation in contact region upon collision may be
 144 important in the description of the adhesive force. In this study, the van der
 145 Waals attractive force is modelled using the Johnson-Kendall-Roberts (JKR)
 146 adhesive model by Johnson et al. [20], which assumes particle deformation in the
 147 contact region to be important when describing the adhesive force. As noted by
 148 Tabor [21], the JKR model is valid when $\lambda_T = (4R\gamma^2/(E^2D_{\min}^3))^{1/3} > 3$, where
 149 γ is the surface energy density, defining half the energy required to separate two
 150 particles in contact and D_{\min} is the minimum separation distance between two
 151 particles, which is commonly assumed to be 1.65 \AA [22, 23].

152 Using the JKR model, the normal contact force upon collision is balanced
 153 by a spring force $\mathbf{F}_{\text{spring},n}$ and an adhesive force $\mathbf{F}_{\text{jkr},n}$:

$$\mathbf{F}_{\text{spring},n} = -\frac{4E}{3R}a^3\mathbf{n} \quad (8)$$

$$\mathbf{F}_{\text{jkr},n} = 4\sqrt{\pi\gamma Ea^3}\mathbf{n} \quad (9)$$

154 where the effective Young's modulus E and effective radius R are given by:

$$\frac{1}{E} = \frac{1-\nu_i^2}{E_i} + \frac{1-\nu_j^2}{E_j}, \quad \frac{1}{R} = \frac{1}{r_i} + \frac{1}{r_j} \quad (10)$$

155 and the contact radius a is the radius of contact area. In equilibrium state where
 156 $\mathbf{F}_{\text{spring}} + \mathbf{F}_{\text{jkr},n} = 0$, the contact radius is $a = (9\pi\gamma R^2/E)^{1/3} = a_0$. When using
 157 DEM, the overlap distance δ_n between particle i and j with positions \mathbf{x}_i and \mathbf{x}_j
 158 is calculated as $\delta_n = r_i + r_j - |\mathbf{x}_i - \mathbf{x}_j|$. For collisions following JKR theory, the
 159 relation between normal overlap δ_n and contact radius a is given by [23, 24]:

$$a^4 - 2R\delta_n a^2 - \frac{4\pi\gamma}{E}R^2 a + R^2\delta_n^2 = 0 \quad (11)$$

160 which in this study is solved using the analytical solution derived by Parteli
 161 et al. [23]. Furthermore, the collision is damped by a normal damping force

162 $\mathbf{F}_{\text{damp},n}$ causing kinetic energy to be dissipated upon impact:

$$\mathbf{F}_{\text{damp},n} = -2\frac{5}{6}\beta\sqrt{S_n m}\mathbf{v}_n \quad (12)$$

163 with effective mass $m^{-1} = m_i^{-1} + m_j^{-1}$, \mathbf{v}_n denoting the relative normal velocity,
 164 β accounting for the kinetic energy lost upon impact through the coefficient of
 165 restitution e and S_n taking the properties of the particles into account:

$$\beta = \frac{\ln(e)}{\sqrt{\ln^2(e) + \pi^2}} \quad (13)$$

$$S_n = 2E\sqrt{R\delta_n} \quad (14)$$

166 where e is material property. Due relatively low collision velocities, plastic
 167 deformation of particle material is not expected to be important. In the
 168 tangential direction, the spring force is given by:

$$\mathbf{F}_{\text{spring},t} = -S_t\Delta s_t \quad (15)$$

169 where Δs_t is the tangential overlap and S_t takes particle properties into account
 170 through $S_t = 8G\sqrt{R\delta_n}$ with effective shear modulus G :

$$\frac{1}{G} = \frac{2 - \nu_i}{G_i} + \frac{2 - \nu_j}{G_j} \quad (16)$$

171 Like in the normal direction, energy is dissipated in the tangential direction,
 172 described by a tangential damping force:

$$\mathbf{F}_{\text{damp},t} = -2\sqrt{\frac{5}{6}}\beta\sqrt{S_t m}\mathbf{v}_t \quad (17)$$

173 where \mathbf{v}_t is the tangential relative velocity. As suggested by Thornton [25],
 174 Thornton and Yin [26], the total tangential force is in the case of JKR adhesion
 175 truncated to fulfil $|\mathbf{F}_{\text{con},t}| \leq \mu_s |F_N + 2F_C|$ with μ_s and F_C being the sliding
 176 friction coefficient and the critical force required to separated agglomerated
 177 particles respectively.

178 2.2.2. Contact torques

179 In the case of adhesive particles, the formation of agglomerates and motion
 180 of particles on a surface tend to be dominated by particles rolling while particles

181 sliding and twisting play negligible roles due to the small particle inertia [27, 28].
 182 As a consequence of the deformed contact region described through JKR theory,
 183 the rolling motion differs from that of non-adhesive particles.

184 In the case of adhesive forces in the contact region between two particles or a
 185 particle and a wall, the point of contact stays behind the centre of mass projected
 186 onto the surface, which results in a torque opposing motion. This rolling
 187 resistance torque acts to obtain the equilibrium condition where the projected
 188 centre of mass and centre of contact are coincident. The rolling resistance torque
 189 is commonly described as proportional to the rolling displacement ξ , which is
 190 found by integrating the rolling velocity \mathbf{v}_r :

$$\xi = \left(\int_{t_0}^{t_1} \mathbf{v}_r(t) dt \right) \cdot \mathbf{t}_r \quad (18)$$

191 where $\mathbf{t}_r = \mathbf{v}_r / |\mathbf{v}_r|$ is the direction of rolling. Based on the instantaneous rolling
 192 displacement ξ , the rolling resistance torque opposing motion is given by [29, 30]:

$$M_r = \begin{cases} k_r \xi & \text{if } \xi < \xi_{\text{crit}} \\ k_r \xi_{\text{crit}} & \text{if } \xi \geq \xi_{\text{crit}} \end{cases} \quad (19)$$

193 where the rolling stiffness is given by $k_r = 4F_C (a/a_0)^{3/2}$. When the particle
 194 is rolled a distance longer than a critical rolling displacement ξ_{crit} , the particle
 195 material slips and a new equilibrium contact region is found. The studies by
 196 Dominik and Tielens [29, 30] suggests that after rolling displacement reaches
 197 a critical value ξ_{crit} , the rolling resistance torque is constant. Based on
 198 experiments, Krijt et al. [31] suggests this critical rolling displacement to be
 199 linked to the equilibrium contact radius a_0 and a material dependent adhesion
 200 hysteresis parameter $\Delta\gamma/\gamma$ through:

$$\xi_{\text{crit}} = \frac{a_0}{12} \frac{\Delta\gamma}{\gamma} \quad (20)$$

201 A value $\Delta\gamma/\gamma = 0.5$ representing a typical value [31] is used throughout this
 202 study.

203 *2.3. Non-dimensional groups*

204 The results are reported based on a set of non-dimensional groups that
 205 govern different aspects of particle transport, particle collision and subsequently
 206 the agglomeration and deposition processes.

207 The numbers governing the transport of particles are the Reynolds number
 208 $Re = U \cdot D/\nu$, the dimensionless particle size $\epsilon = d_p/D$, the particle to fluid
 209 density ratio $\chi = \rho_p/\rho_f$, the Stokes number $St = \rho_p d_p^2 U / (18\mu D) = \chi \epsilon^2 Re / 18$
 210 and the particle volume fraction defining the volume occupied by particles
 211 in the fluid domain $\phi = \sum_{n=1}^N V_p / (\sum_{n=1}^N V_p + V_f)$. The importance of gravity is
 212 described through the Froude number $Fr = U / \sqrt{g_r d_p}$, where $g_r = (1 - 1/\chi)g$ is
 213 the buoyancy corrected gravity acceleration.

214 Likewise, the collision and agglomeration processes are governed by a set
 215 of non-dimensional groups. The stiffness of a common collision is described by
 216 making the effective Young's modulus dimensionless using the particle density,
 217 bulk velocity and particle density forming $\lambda = E / (\rho_p U^2)$. Likewise, to describe
 218 the sticking behaviour upon impact, the surface energy density, describing the
 219 strength of the adhesive force, is made non-dimensional by the particle density,
 220 bulk velocity and particle diameter, forming $Ad = \gamma / (\rho_p U^2 d_p)$.

221 Due to stiffness of most common materials, the time step size required
 222 to resolve particle collisions is typically in the order of nano seconds. A
 223 common approach to deal with the low time step sizes is to reduce the particle
 224 stiffness several orders of magnitude making collisions take place over longer time
 225 periods. For non-adhesive particles colliding without any plastic deformation,
 226 the rebound velocity is solely a function of velocity before collision and the
 227 coefficient of restitution. However, for collisions involving adhesive particles,
 228 the reduced particle stiffness has to be balanced by a reduction in adhesive
 229 inter-molecular force so that the collision outcome remains the same. In this
 230 study, the high particle stiffness is reduced by decreasing the effective Young's

231 modulus while modifying the surface energy density as:

$$\gamma_{\text{mod}} = \gamma \left(\frac{E_{\text{mod}}}{E} \right)^{2/5} \quad (21)$$

232 or in terms of the dimensionless elasticity parameter λ and adhesiveness
233 parameter Ad:

$$\text{Ad}_{\text{mod}} = \text{Ad} \left(\frac{\lambda_{\text{mod}}}{\lambda} \right)^{2/5} \quad (22)$$

234 which is shown by Hærvig et al. [11] to make the collision outcome independent
235 of a reduction in Young’s modulus. For particles colliding in a viscous fluid,
236 the fluid being forced away in the contact prior to collision may have a non-
237 negligible effect on the rebound velocity [32, 33]. This importance of this effect is
238 described through the collision Stokes number $\text{St}^* = (m_p + C_M m_f) v / (6\pi\mu r_p)$,
239 where $C_M = 0.5$ is the added mass coefficient for spheres and v denotes the
240 velocity before being affected by the surface. For solid particles colliding with
241 $\text{St}^* < 10$, Legendre et al. [34] suggests all energy to be dissipated while for
242 higher values of St^* , this effect quickly becomes negligible.

243 *2.4. Particle-fluid interaction*

244 The fluid force governing the transport of particles $\mathbf{F}_{\text{fluid}}$ in equation (6)
245 is obtained by point-force approximations due to the large number of particles
246 involved. The fluid force is split up into different contributions that include: a
247 fluid drag force, Saffman lift force due to local shear in the flow field, Magnus
248 lift force due to local relative angular velocity between particle and fluid, a
249 buoyancy-corrected gravity force, a added mass force due to acceleration of
250 nearby fluid, a Basset history force due to delay in boundary layer build-up and
251 a Brownian motion force due to random collisions with fluid molecules.

252 Due to the large density ratio between particles and air, typically
253 $\chi = \rho_p / \rho_f > 1000$, the added mass and history forces are neglected in the present
254 study as suggested by Dritselis [35], Armenio and Fiorotto [36].

255 Also, gravity plays a minor role compared to drag for the small particles
256 considered. As shown by Marshall [37], the ratio of gravity to Stokes drag can be

257 approximated by combining the dimensionless particle diameter and the Froude
 258 number as: $F_g/F_d = \mathcal{O}(\epsilon \text{Fr}^2)^{-1}$. For particles investigated in this study, this
 259 ratio is $F_g/F_d = \mathcal{O}(10^{-2})$ and thus gravity is expected to play a negligible
 260 role. Furthermore, molecular effects due to Brownian motion are neglected
 261 as the particles considered are several orders of magnitude larger than fluid
 262 molecules. Moreover, it is assumed that the mean free path of fluid molecules is
 263 much smaller than the particle size ($\text{Kn} \ll 1$), so that the standard continuum
 264 assumption is valid and the effect of surface slip is negligible.

265 *2.4.1. Fluid force contributions*

266 As inter-particle van der Waals forces cause agglomerates to form and
 267 particles to deposit, the particle volume fraction α_p increases locally. When
 268 particles agglomerate, the individual particles in the agglomerate are affected by
 269 fluid forces that are significantly different from the fluid force on single particles
 270 in dilute flows. Therefore, a drag formulation taking the presence of surrounding
 271 particles into account is used. In this study, the Lattice-Boltzmann-based drag
 272 formulation suggested by Hill et al. [14, 15] and later modified by Benyahia et al.
 273 [38] to cover particle Reynolds numbers up to 100 and particle volume fractions
 274 approaching the closed-pack solution is used.

275 For particles in the viscous sub-layer, where strong velocity gradients exist,
 276 Saffman lift force is expected to be important as suggested by McLaughlin [39].
 277 Furthermore, Magnus lift due to relative angular velocity between particle and
 278 fluid is included as off-centre collisions may result in particles rebounding with
 279 non-negligible angular velocities. To model these fluid force contributions, the
 280 expressions derived by Kurose and Komori [40] and McLaughlin [41] are used.

281 **3. Results and validation**

282 *3.1. Fluid domain and discretisation*

283 To make the results independent of stream-wise boundaries, the fluid domain
 284 is made periodic with a length longer than the elongated coherent turbulence

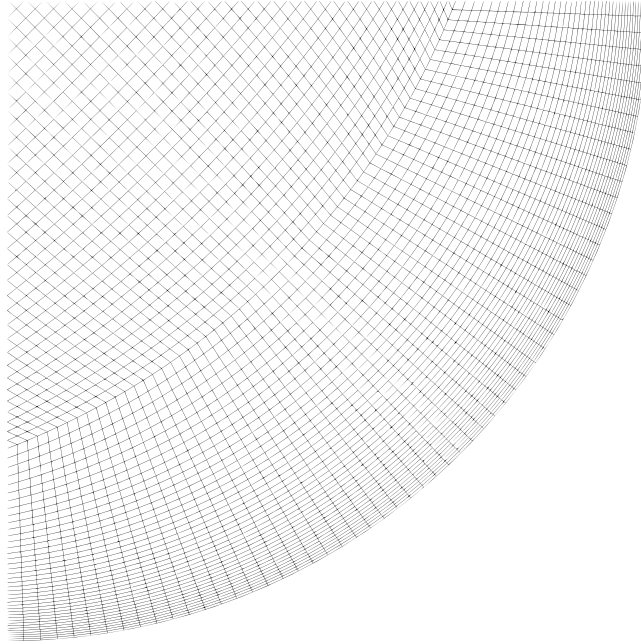


Figure 2: Mesh topology visualised by a quarter of the cross-sectional plane.

285 structures in the boundary layer extending around $L_x^+ \approx 1000$ [42]. In this
 286 study, a domain length of $L/D = 4$, corresponding to $L_x^+ \approx 2500$ in viscous
 287 units, is chosen to make sure no coherent structures extend throughout the
 288 domain. Thus, the turbulence statistics are not affected by the size of the
 289 computational domain. A quarter of the cross-sectional mesh topology is shown
 290 in figure 2. As the agglomeration formation process takes place over stream-
 291 wise distances significantly longer than $L/D = 4$, the DEM domain is made
 292 periodic as well. In that way, the overall particle volume fraction ϕ stays
 293 constant throughout the simulation. This approach gives detailed information
 294 on the mechanisms governing agglomeration and deposition and how changes in
 295 fluid/particle properties affect early stages of agglomeration and deposition.
 296 Later stages of the deposition process where bridges and layers of multiple
 297 particles form, see figure 1, would require particles to be added throughout the
 298 simulation or a significantly higher initial particle concentration, which would
 299 alter the early stages of agglomeration and deposition processes.

Table 1: Details of the three different meshes with θ , r and x denoting circumferential, radial and axial directions respectively. The mesh topology is shown in figure 2.

Resolution	(N_θ, N_r, N_x)	$\Delta\theta_{\max}^+$	Δx_{\max}^+
Coarse	(160,90,160)	12	15
Medium	(200,110,200)	10	12.5
Fine	(240,140,240)	8	10

300 *3.2. Validation of flow field*

301 To validate the statistics of the flow field without particles added, the
 302 boundary layer velocity profile is compared for the different meshes listed in
 303 table 1. The various grid resolutions are compared to experiments by den
 304 Toonder and Nieuwstadt [43] and boundary layer theory. Parameters for
 305 the different meshes are listed in table 1. In this section, the time-averaged
 306 turbulent flow obtained by LES simulations without particles is compared
 307 to theory to ensure reasonable accuracy compared to experimental data and
 308 theory. The time-averaged flow field is found by averaging over a time period
 309 of $t^+ = t \cdot U/D = 400$ after which the mean flow field statistics are found
 310 to be independent of time. Figure 3 shows how the numerical simulations
 311 compare to boundary layer theory and experiments at $\text{Re} = 10,000$. As
 312 seen in figure 3, the more resolving LES simulations approach the experiment
 313 by den Toonder and Nieuwstadt [43].

314 *3.3. Particle properties*

315 Particles with $d_p = 10 \mu\text{m}$ and $\rho_p = 2500 \text{ kg/m}^3$ are transported in pipe flow
 316 with $D = 40 \text{ mm}$ and mean velocity $U = 5 \text{ m/s}$ so that $\text{Re} = U \cdot D/\nu \approx 10,000$,
 317 $\text{St} = \rho_p d_p^2 U / (18\mu D) \approx 0.1$ and $\epsilon = d_p/D = 0.25 \cdot 10^{-3}$.

318 The particles are affected by an inter-particle adhesive force characterised
 319 by a surface energy density and develop a flattened contact region as predicted
 320 by JKR theory. The particles have stiffness and adhesiveness such that
 321 $\lambda = E/(\rho_p U^2) \approx 500 \cdot 10^3$, $\text{Ad} = \gamma/(\rho_p U^2 d_p) \approx 0.08$ and coefficient of

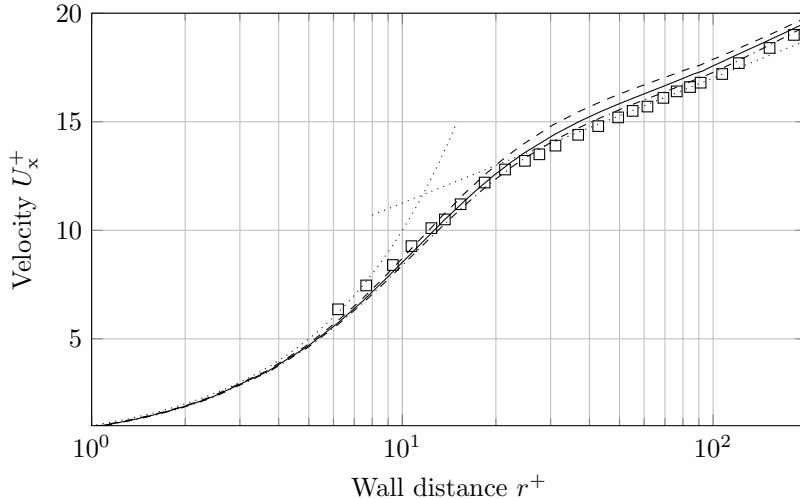


Figure 3: Time-averaged turbulent boundary layer velocity profile for $Re = UD/\nu = 10,000$ for different mesh resolutions, see table 1: (---) Coarse resolution LES; (—) Medium resolution LES; (-·-·-) Fine resolution LES; (·····) Viscous sub-layer with $U_x^+ = r^+$ and log-law layer with $U_x^+ = 2.5 \ln(r^+) + 5.5$; \square Experiment for $Re = 10,000$ by den Toonder and Nieuwstadt [43].

322 restitution $e \approx 0.3$. The particles are initially placed randomly throughout
 323 the turbulent flow without any particles being in contact. Instead of using a
 324 more or less sophisticated correlation to prescribe the initial velocity of particles,
 325 the particles are inserted with zero velocity in the present study. A constant
 326 volume fraction $\phi = 10^{-3}$ is used in all simulations.

327 However, due to the significant difference in particle to pipe size ratio
 328 $\epsilon = d_p/D = 0.25 \cdot 10^{-6}$, it takes $\approx 0.4 \cdot 10^9$ particles to get a volume fraction
 329 $\phi = 0.001$, making the simulations practically computationally impossible for
 330 the DEM method. Instead, the particle size is increased from $\epsilon = 0.25 \cdot 10^{-3}$
 331 to $\epsilon = 5 \cdot 10^{-3}$, while scaling the other non-dimensional groups to get the
 332 parameters listed above. For example, to ensure the artificially larger particles
 333 have the same particle to fluid response time, the density ratio χ is scaled
 334 according to $St = \chi \epsilon^2 Re / 18$. Furthermore, as the adhesiveness parameter is a
 335 function of particle diameter, the surface energy density γ is scaled as well so

336 that the adhesive behaviour remains the same despite an increase in particle
337 size.

338 *3.4. Overview of simulations*

339 The following gives an overview of the different simulations carried out in
340 the present study. The simulation parameters are summarised in terms of non-
341 dimensional numbers in table 2.

342 *Simulation no. 1–21.* Depending on the Stokes number, the particles will be
343 affected by different turbulence scales. As the Stokes number approaches zero,
344 the particles will be affected by increasingly smaller eddies. In this study where
345 LES is used, it is important that particles are unaffected by the eddies not
346 being resolved. Therefore, simulations with varying Stokes number $St = 0.4 \cdot 2^x$,
347 $x = 0 \dots 6$ and carried out using the three meshes listed in table 1.

348 *Simulation no. 22–28.* As already mentioned, the agglomeration processes
349 considered in this study have elasticity parameter $\lambda \approx 500 \cdot 10^3$. Using this
350 value, particle-particle and particle-wall collisions happen over time intervals
351 in the order of nano seconds. To reduce computational time, a modified
352 elasticity parameter is introduced: $\lambda_{\text{mod}} = \lambda \cdot 0.001 = 500$. By reducing the
353 elasticity parameter, the particles behave more adhesive. Therefore, to ensure
354 the collision outcome stays independent of reduction in elasticity parameter,
355 the criterion in (22) is used to reduce the adhesiveness parameter so that
356 $Ad_{\text{mod}} = 0.08 \cdot 0.001^{2/5} = 0.005$. However, as noted in the study, introducing
357 a lower adhesiveness parameter also reduces the critical force $F_C = 3\pi R\gamma$
358 required to separate two agglomerated particles. Therefore, simulations 22–28
359 are carried out with $\lambda_{\text{mod}} = \lambda/100$ instead of $\lambda_{\text{mod}} = \lambda/1000$ to ensure
360 the agglomeration rate is in fact independent of a reduction in elasticity and
361 adhesiveness parameter.

362 *Simulation no. 29–42.* The purpose of simulation 29–42 is to investigate the
363 transition from particles acting almost non-adhesive to particles being highly

Table 2: Details on the simulations in terms of non-dimensional groups: Flow Reynolds number $Re = U \cdot D/\nu$, particle size $\epsilon = d_p/D$, Stokes number $St = \chi\epsilon^2 Re/18$, elasticity parameter $\lambda = E/(\rho_p U^2)$, adhesiveness parameter $Ad = \gamma/(\rho_p U^2 d_p)$, coefficient of restitution e , particle volume fraction ϕ and LES resolution (see table 1). The integer x is used to describe a range of simulation parameters.

No.	x	Re	ϵ	St	λ	Ad	e	ϕ	LES res.
1–7	0..6	10^4	0.005	$0.4 \cdot 2^x$	500	0.05	0.3	10^{-3}	Coarse
8–14	0..6	10^4	0.005	$0.4 \cdot 2^x$	500	0.05	0.3	10^{-3}	Medium
15–21	0..6	10^4	0.005	$0.4 \cdot 2^x$	500	0.05	0.3	10^{-3}	Fine
22–28	0..6	10^4	0.005	$0.4 \cdot 2^x$	5000	$0.05 \cdot 10^{2/5}$	0.3	10^{-3}	Medium
29–35	0..6	10^4	0.005	1	500	$0.001 \cdot 2^x$	0.3	10^{-3}	Medium
36–42	0..6	10^4	0.005	10	500	$0.001 \cdot 2^x$	0.3	10^{-3}	Medium

364 adhesive at constant Stokes numbers. For this purpose, the adhesive parameter
365 is varied in the range 0.001 to 0.064 at a constant Stokes number $St = 1$
366 (simulations 29–35). Similar simulations are carried out for $St = 10$ (simulations
367 36–42).

368 3.5. Effect of sub-grid scale turbulence eddies on particle agglomeration

369 As the smallest eddy scales are modelled by a sgs-model, it is important
370 that these unresolved scales do not affect particle motion and subsequently
371 agglomeration and deposition. Ultimately, as particles with sufficiently low
372 response time, e.g. tracer particles, would respond to all turbulence scales,
373 a DNS solution would be required in this case. However, as pointed out by
374 Armenio [44], particles with finite mass do not respond to smaller turbulence
375 scales, suggesting LES to be an appreciate method to investigate particles
376 interactions in a turbulent flow. Figure 4 shows how the agglomeration rate
377 by the average number of particles in each agglomerate as function of time for
378 different particle Stokes numbers. As the figure shows, particles with $St = \mathcal{O}(10)$
379 are almost unaffected by changes in mesh resolution. This suggests that these
380 particles are unaffected by eddies smaller than the grid size of the coarse mesh.
381 For particles with $St = \mathcal{O}(1)$, there is pronounced difference between the coarse
382 and medium mesh resolutions, suggesting particles with $St = \mathcal{O}(1)$ to respond to

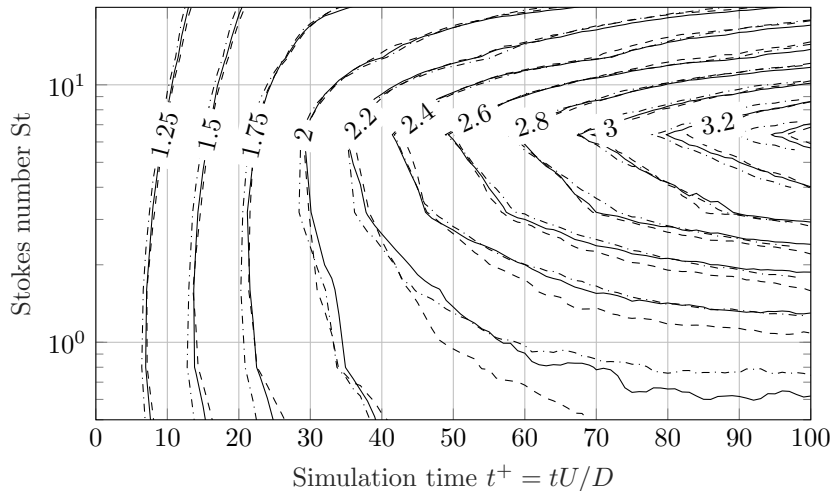


Figure 4: Effect of mesh resolution, see table 1, and Stokes number on particle agglomeration behaviour visualised by the average number of particles per agglomerate. As no particles are in contact at $t^+ = 0$, the average number of particles is 1. See simulation no. 1–21 in table 2 for simulation parameters: (---) Coarse resolution LES; (—) Medium resolution LES; (-·-·-) Fine resolution LES.

383 eddies not being resolved by the coarsest mesh. The difference is insignificant
 384 when comparing the medium and fine mesh resolutions, suggesting that the
 385 particles are almost unaffected by the smaller eddies not being resolved by
 386 the medium resolution mesh. As a consequence, the medium resolution mesh
 387 from table 1 is used to investigate how changes particle response time and
 388 adhesiveness affect the agglomeration rate.

389 3.6. Effect of introducing softer particles

390 In the present study, the particle stiffness is reduced by a factor 1000
 391 from $\lambda = 500 \cdot 10^3$ to $\lambda_{\text{mod}} = 500$ in order to increase collision duration
 392 and consequently allow for an increased DEM time step size. The result is
 393 smaller difference between fluid time step size δt_f and particle time step size
 394 δt_{col} that decrease the computational time. In order for the collision outcome
 395 (sticking/rebounding) to remain the same despite a reduced particle stiffness,
 396 the particle adhesiveness is modified using the criterion given in (22), giving a

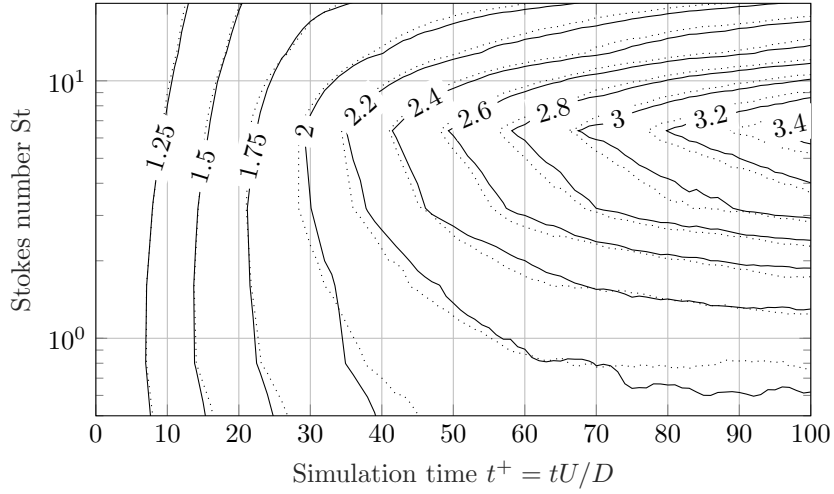


Figure 5: Effect of introducing softer particles using the criterion in (22). Contours show particle agglomeration rate visualised by the average number of particles contained in each agglomerate for particle elasticity parameters $\lambda = 500$ and $\lambda = 5000$. See simulation no. 8–14 and 22–28 in table 2 for simulation parameters: (—) $\lambda_1 = 500$, $\text{Ad}_1 = 0.05$; (⋯⋯) $\lambda_2 = 5000$, $\text{Ad}_2 = \text{Ad}_1 \cdot (\lambda_2/\lambda_1)^{2/5}$.

397 modified collision duration $\Delta t_{\text{col,mod}} = \Delta t_{\text{col}} (\lambda/\lambda_{\text{mod}})^{2/5}$ [11]. To ensure that
 398 the agglomeration process is in fact independent of this change, an additional
 399 set of simulations with $\lambda = 5000$ is carried out. Figure 5 shows results for
 400 $\lambda_1 = 500$ and $\lambda_2 = 5000$ with adhesiveness parameters $\text{Ad}_1 = 0.05$ and
 401 $\text{Ad}_2 = \text{Ad}_1 (\lambda_2/\lambda_1)^{2/5}$. As figure 5 shows, the agglomeration process is almost
 402 independent of a change in elasticity parameter from $\lambda = 500$ to $\lambda = 5000$ when
 403 reducing the adhesiveness parameter by (22) for particles with Stokes numbers
 404 in the range $\text{St} = 0.4$ to $\text{St} = 25.6$.

405 3.7. Effect of particle Stokes number on agglomeration and deposition

406 Depending on the Stokes number, the agglomeration process is governed by
 407 different distinct mechanisms. At sufficiently low Stokes numbers $\text{St} \ll 1$, the
 408 particles behave as tracer particles and respond to all turbulence scales. That is,
 409 collisions happen due to the finite size of particles that all follow different fluid
 410 streamlines. At sufficiently high Stokes numbers $\text{St} \gg 1$, the particle velocity is

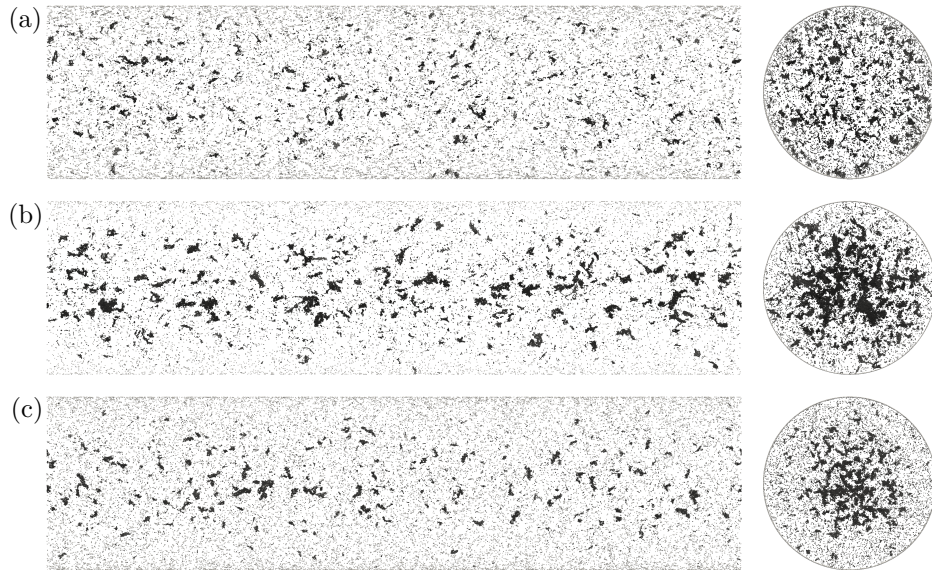


Figure 6: Agglomerating and depositing behaviour at time $t^+ = tU/D = 100$: (a) $St = 0.8$, see simulation no. 9 in table 2; (b) $St = 6.4$, see simulation no. 12 in table 2; (c) $St = 25.6$, see simulation no. 14 in table 2. To distinguish between moving particles and deposited particles, the particles are coloured according to their velocity magnitude. White corresponds to low velocity and black corresponds to high velocity.

411 only weakly correlated with the local fluid velocity. Figure 6 gives an overview
 412 of the agglomerates formed at time $t^+ = 100$ for Stokes numbers of 0.8, 6.4 and
 413 25.6. As shown in figure 6, the agglomeration rate is highest at intermediate
 414 Stokes numbers where larger agglomerates are being formed in the centre of the
 415 pipe, see 6(b). This can be explained by the increased collision frequency when
 416 the accelerative-correlated collision mechanism is dominant. Figure 7 gives an
 417 overview of the agglomeration process in terms of particle location. Figure 7
 418 shows the radial concentration of particles in terms of local volume fraction ϕ_r
 419 normalised by the overall particle volume fraction ϕ in different radial intervals
 420 r normalised by the pipe radius R at different times for simulation no. 9 in table
 421 2. As shown in figure 7, there is an increasing amount of particles sticking to
 422 the wall, while the concentration profile throughout the pipe remains relatively
 423 constant. This can be explained by the low Stokes number where $\tau_p < \tau_f$ causing

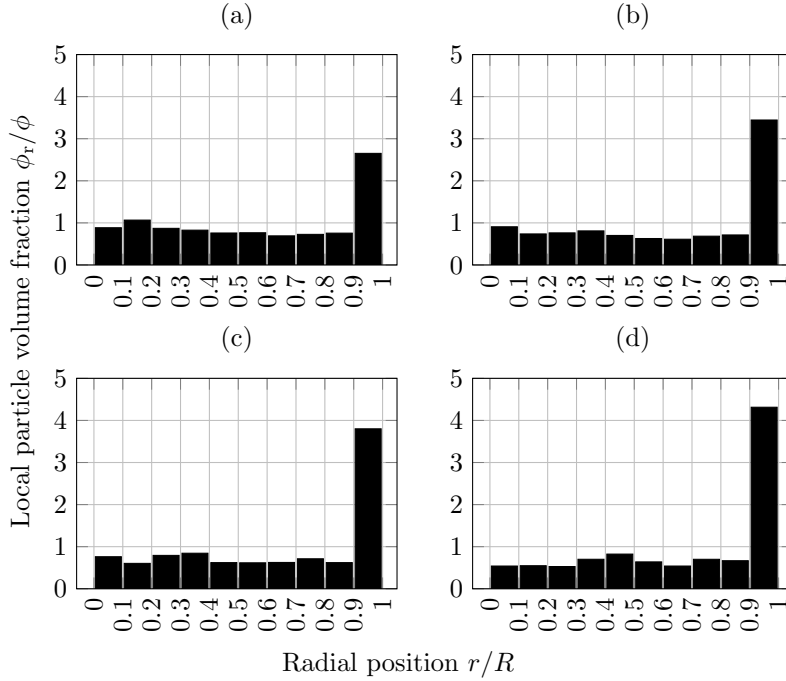


Figure 7: Local particle volume fraction ϕ_r in radial intervals of $r/R = 0.1$ normalised by the overall particle volume fraction ϕ at different times. Stokes number $St = 0.8$, see simulation no. 9 in table 2 for simulation parameters: (a) $t^+ = tU/D = 25$; (b) $t^+ = tU/D = 50$; (c) $t^+ = tU/D = 75$; (d) $t^+ = tU/D = 100$.

424 particles to respond to most of the eddies resolved in turbulent flow. Figure 8
 425 shows how the particle concentration profile is affected by an increase in Stokes
 426 number. At a slightly higher Stokes number $St = 6.4$, particle accumulate
 427 in the centre of the pipe and near the wall. This phenomenon is explained
 428 by an increased collision frequency, causing agglomerates to form quickly. As
 429 agglomerates are formed, the effective response time for agglomerates is greatly
 430 increased, causing agglomerates to drift towards the centre of the pipe due
 431 to the shear velocity profile. Figure 9 shows the radial particle concentration
 432 profile for higher Stokes number of $St = 25.6$. At a higher Stokes number, the
 433 primary particles are expected to almost unaffected by turbulent fluctuations
 434 so that the average flow field cause particles to drift towards the centre of the
 435 pipe. However, compared to the lower Stokes number in figure 7, more particles

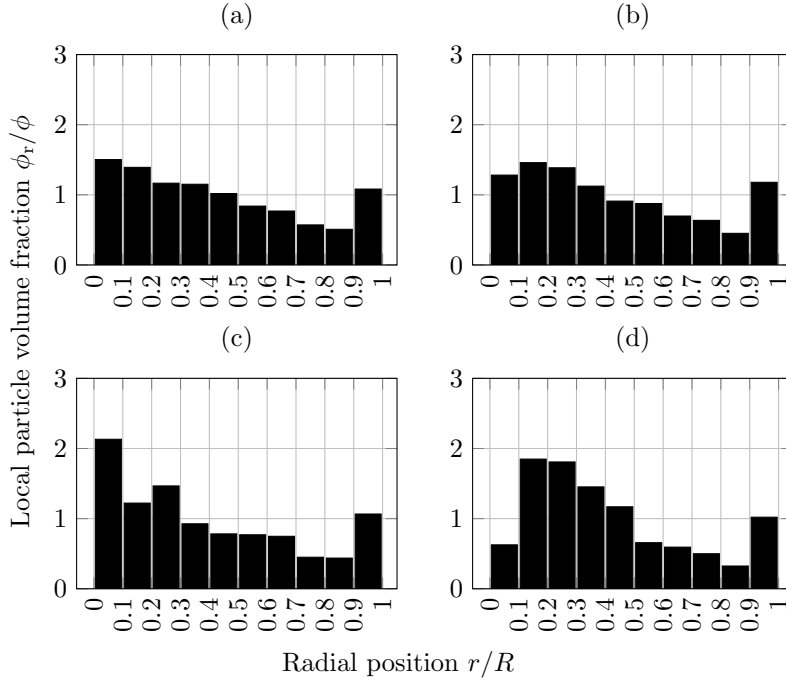


Figure 8: Local particle volume fraction ϕ_r in radial intervals of $r/R = 0.1$ normalised by the overall particle volume fraction ϕ at different times. Stokes number $St = 6.4$, see simulation no. 12 in table 2 for simulation parameters: (a) $t^+ = tU/D = 25$; (b) $t^+ = tU/D = 50$; (c) $t^+ = tU/D = 75$; (d) $t^+ = tU/D = 100$.

436 adhere to the surface of the pipe. This can be explained by the particles not
 437 responding as quickly to the high velocity gradient close to the wall, causing
 438 particles to move through the boundary layer and adhere to the wall.

439 3.8. Effect of particle adhesiveness on agglomeration

440 In the following, the effect of increased particle adhesiveness is investigated.
 441 The surface energy density γ is varied, resulting in adhesiveness parameters
 442 in the range $Ad = 0.001$ to $Ad = 0.064$, covering particles that range from
 443 weakly to highly adhesive. Figure 10 gives an overview of how the adhesiveness
 444 parameter changes the agglomerating behaviour for particles with $St = 1$. As
 445 figure 10(a) shows, the weakly adhesive particles only form smaller agglomerates
 446 in the centre of the tube and are significantly more uniformly distributed than

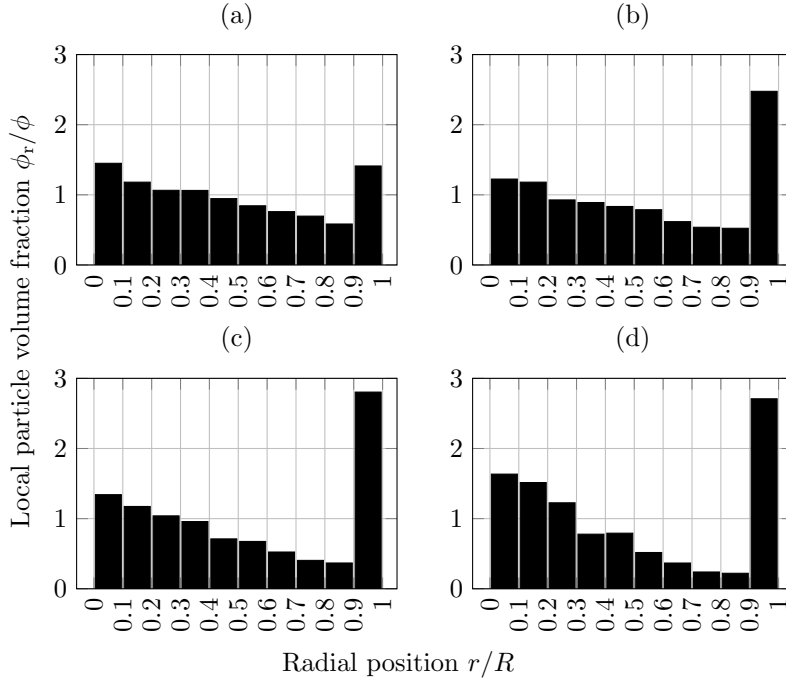


Figure 9: Local particle volume fraction ϕ_r in radial intervals of $r/R = 0.1$ normalised by the overall particle volume fraction ϕ at different times. Stokes number $St = 25.6$, see simulation no. 14 in table 2 for simulation parameters: (a) $t^+ = tU/D = 25$; (b) $t^+ = tU/D = 50$; (c) $t^+ = tU/D = 75$; (d) $t^+ = tU/D = 100$.

447 the more adhesive particles in figure 10(b) and 10(c). Figure 11 shows the
 448 average number of particles per agglomerate as function of simulation time at
 449 different adhesiveness parameters. As figure 11 shows, the weakly adhesive
 450 particles with $Ad = 0.001$ are in average contained in agglomerates consisting
 451 of less than 1.025 particles. Even for non-adhesive particles with $Ad = 0$,
 452 a non-zero average agglomeration size is expected due to the finite collision
 453 duration, causing particles to be in contact for short durations before they
 454 separate again due to repulsive contact forces. At slightly higher adhesiveness
 455 parameters, significant agglomeration is observed with agglomerates being
 456 present throughout the flow field. Figure 12 shows how an increase in
 457 adhesiveness parameter from $Ad = 0.002$ to $Ad = 0.064$ affects the local particle
 458 concentration. As depicted in figure 12, an increasing amount of particles adhere

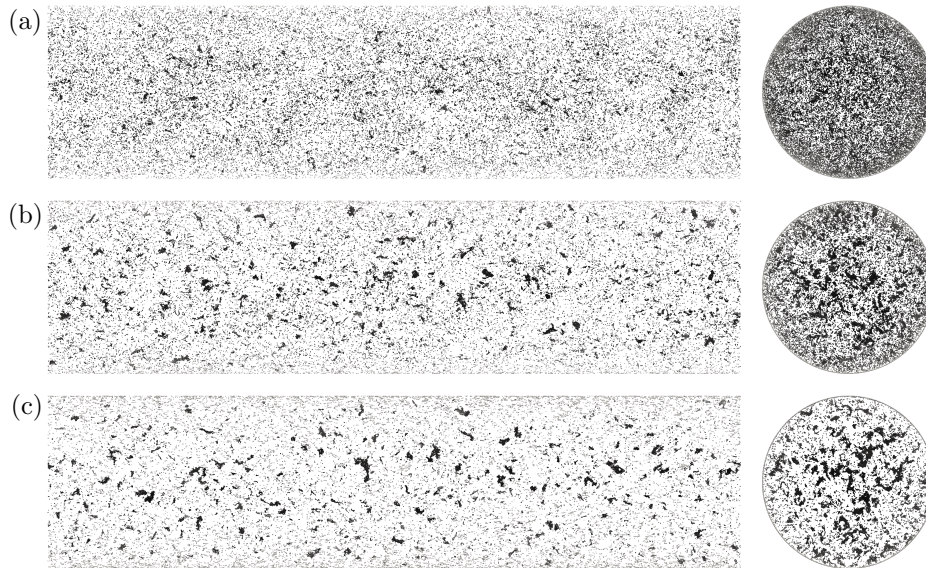


Figure 10: Overview of agglomerating behaviour of mono-dispersed particles with different adhesiveness parameters at time $t^+ = tU/D = 100$: (a) $Ad = 0.016$; (b) $Ad = 0.032$; (c) $Ad = 0.064$. To distinguish between moving particles and deposited particles, the particles are coloured according to their velocity. White corresponds to low velocity and black corresponds to high velocity.

459 to the wall when the adhesiveness parameter is increased.

460 **4. Conclusions and discussion**

461 Numerical results of how micron-sized agglomerate and deposit in a periodic,
 462 fully-developed turbulent pipe flow with $Re = 10000$ flow are presented. In
 463 this study, focus is on the first stages of agglomeration and deposition up to
 464 $t^+ = tU/D = 100$.

465 Firstly, Large Eddy Simulations (LES) are compared to experiments from
 466 literature in terms of boundary layer profile. With particles added, the effect of
 467 sub-grid scale fluctuations are linked to the Stokes number to ensure particle
 468 motion is independent of the unresolved velocity fluctuations. Fluid-particle
 469 interactions are based on the point-particle approach while particle-particle
 470 and particle-wall interactions are resolved directly using the soft-sphere DEM

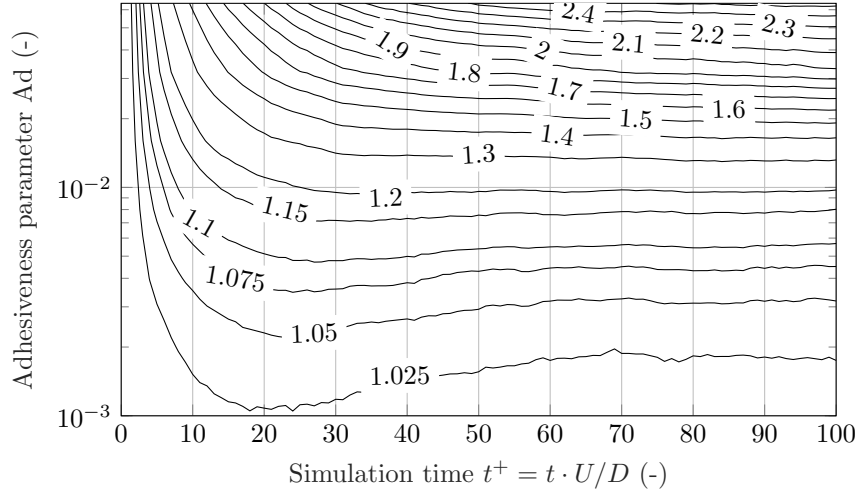


Figure 11: Effect of particle adhesiveness parameter $Ad = \gamma/(\rho_p U^2 d_p)$ on particle agglomeration behaviour visualised by the average number of particles in each agglomerate, see simulation no. 29–35 in table 2 for simulation parameters.

471 approach relying on the physical properties of the particles, where the adhesive
 472 behaviour is described through JKR theory.

473 Secondly, the effects of changes in particle response time on the
 474 agglomeration and deposition rates are investigated. By varying the Stokes
 475 number from 0.4 to 25.6, the results show a peak in agglomeration rate at
 476 $St = 6.4$, which can be explained by a high collision frequency. For a low
 477 Stokes number of $St = 0.8$, agglomerates are found throughout the flow field.
 478 Due to low particle response time, the particle concentration is close to uniform
 479 throughout the tube with a increase in particle concentration near the wall due
 480 to particle being captured by the wall. At a higher Stokes number $St = 6.4$, the
 481 particles tend to accumulate in the centre of the tube. This is expected to be
 482 caused by the high collision rate that quickly form agglomerates that increase
 483 the effective particle size so that the Saffman lift force dominates their radial
 484 motion causing them to move towards the centre of the pipe. At an even higher
 485 Stokes number $St = 25.6$, the particles are observed to accumulate in the centre
 486 of the pipe as well.

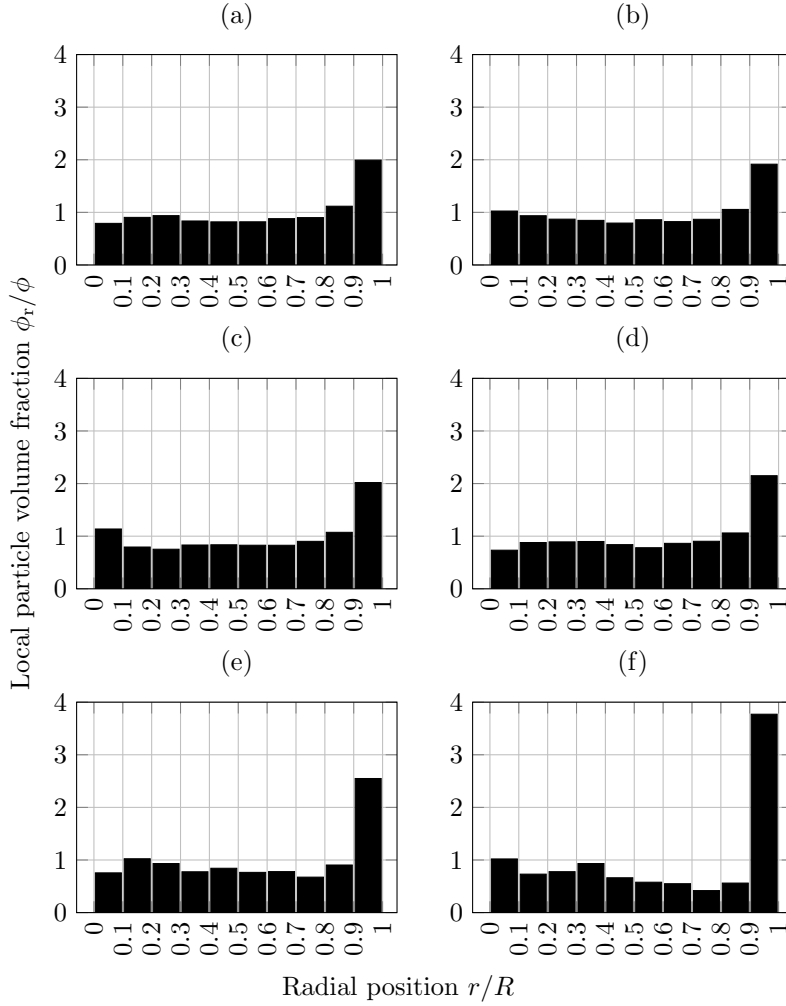


Figure 12: Local particle volume fraction ϕ_r in radial intervals of $r/R = 0.1$ normalised by the overall particle volume fraction ϕ at $t^+ = 100$. Varying adhesiveness parameter, see simulation no. 30–35 in table 2 for simulation parameters: (a) $Ad = 0.002$; (b) $Ad = 0.004$; (c) $Ad = 0.008$; (d) $Ad = 0.016$; (e) $Ad = 0.032$; (f) $Ad = 0.064$.

487 Thirdly, The adhesiveness parameter $Ad = \gamma/(\rho_p U^2 d_p)$ is varied from 0.001
 488 to 0.064 resulting in particles that behave only weakly adhesive to highly
 489 adhesive. Regardless of adhesiveness parameter, the particle concentration
 490 throughout the flow field is observed to be more or less uniform with a peak at
 491 the wall that increases with adhesiveness parameter due to particles adhering

492 to the wall.

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References

- [1] P. Saffman, J. Turner, On the collision of drops in turbulent clouds, *Journal of Fluid Mechanics* 1 (1956) 16–30, DOI: <http://doi.org/10.1017/S0022112056000020>.
- [2] J. Abrahamson, Collision rates of small particles in a vigorously turbulent fluid, *Chemical Engineering Science* 30 (1975) 1371–1379, DOI: [http://doi.org/10.1016/0009-2509\(75\)85067-6](http://doi.org/10.1016/0009-2509(75)85067-6).
- [3] C. Meyer, D. Deglon, Particle collision modeling - A review, *Minerals Engineering* 24 (2011) 719–730, DOI: <http://doi.org/10.1016/j.mineng.2011.03.015>.
- [4] B. K. Brunk, D. L. Kock, L. W. Lion, Turbulent coagulation of colloidal particles, *Journal of Fluid Mechanics* 364 (1998) 81–113, DOI: <http://doi.org/10.1017/S0022112098001037>.
- [5] B. K. Brunk, D. L. Kock, L. W. Lion, Observations of coagulation in isotropic turbulence, *Journal of Fluid Mechanics* 371 (1998) 81–107, DOI: <http://doi.org/10.1017/S0022112098001037>.
- [6] A. Brasil, T. Farias, M.G.Carvalho, U. Koylu, Numerical Characterization of the morphology of aggregated particles, *Aerosol Science* 32 (2001) 489–508, DOI: [http://doi.org/10.1016/S0021-8502\(00\)00097-5](http://doi.org/10.1016/S0021-8502(00)00097-5).

- [7] M. Sommerfeld, S. Stübing, A novel Lagrangian agglomerate structure model, *Powder Technology* 319 (2017) 34–52, DOI: <http://dx.doi.org/10.1016/j.powtec.2017.06.016>.
- [8] J. Marshall, Particle aggregation and capture by walls in a particulate aerosol channel flow, *Aerosol Science* 38 (2007) 333–351, DOI: <http://dx.doi.org/j.jaerosci.2007.01.004>.
- [9] T. Kobayashi, T. Tanaka, N. Shimada, T. Kawaguchi, DEM-CFD analysis of fluidization behavior of Geldart Group A particles using a dynamic adhesion force model, *Powder Technology* 248 (2013) 143–152, DOI: <http://dx.doi.org/10.1016/j.powtec.2013.02.028>.
- [10] Y. Gu, A. Ozel, S. Sundaresan, A modified cohesion model for CFD-DEM simulations of fluidization, *Powder Technology* 296 (2016) 17–28, DOI: <http://dx.doi.org/10.1016/j.powtec.2015.09.037>.
- [11] J. Hærvig, U. Kleinhans, C. Wieland, H. Spliethoff, A. L. Jensen, K. Sørensen, T. J. Condra, On the Adhesive JKR Contact and Rolling Models for Reduced Particle Stiffness Discrete Element Simulations, *Powder Technology* 319 (2017) 472–482, DOI: <http://doi.org/10.1016/j.powtec.2017.07.006>.
- [12] F. D. Dizaji, J. S. Marshall, On the Significance of Two-Way Coupling in Simulation of Turbulent Particle Agglomeration, *Powder Technology* 318 (2017) 83–94, DOI: <http://doi.org/10.1016/j.powtec.2017.05.027>.
- [13] S. Ergun, A. A. Orning, Fluid Flow through Randomly Packed Columns and Fluidized Beds, *Industrial & Engineering Chemistry* 41 (1949) 1179–1184, DOI: <http://doi.org/10.1021/ie50474a011>.
- [14] R. J. Hill, D. L. Koch, A. J. Ladd, The first effects of fluid inertia on flows in ordered and random arrays of spheres, *Journal of Fluid Mechanics* 448 (2001) 213–241, DOI: <http://dx.doi.org/10.1017/S0022112001005948>.

- [15] R. J. Hill, D. L. Koch, A. J. Ladd, Moderate-Reynolds-number flows in ordered and random arrays of spheres, *Journal of Fluid Mechanics* 448 (2001) 243–278, DOI: <https://doi.org/10.1017/S0022112001005936>.
- [16] M. Dietzel, M. Sommerfeld, Numerical calculation of flow resistance for agglomerates with different morphology by the Lattice-Boltzmann Method, *Powder Technology* 250 (2013) 122–137, DOI: <http://doi.org/10.1016/j.powtec.2013.09.023>.
- [17] M. Afkhami, A. Hassanpour, M. Fairweather, D. Njobuenwu, Fully coupled LES-DEM of particle interaction and agglomeration in a turbulent channel flow, *Computers and Chemical Engineering* 78 (2015) 24–38, DOI: <http://dx.doi.org/10.1016/j.compchemeng.2015.04.003>.
- [18] F. Nicoud, F. Ducros, Subgrid-scale Stress Modelling Based on the Square of the Velocity Gradient Tensor, *Flow, Turbulence and Combustion* 62 (1999) 183–200, DOI: <http://dx.doi.org/10.1023/A:1009995426001>.
- [19] P. Cundall, O. Strack, A discrete numerical model for granular assemblies, *Géotechnique* 29 (1979) 47–65, DOI: <http://dx.doi.org/10.1680/geot.1979.29.1.47>.
- [20] K. L. Johnson, K. Kendall, A. D. Roberts, Surface energy and the contact of elastic solids, *Proc. R. Soc. Lond.* 324 (1971) 301–313, DOI: <http://dx.doi.org/10.1098/rspa.1971.0141>.
- [21] D. Tabor, Surface Forces and Surface Interactions, *Journal of Colloid and Interface Science* 58 (1975) 2–13, DOI: [http://dx.doi.org/10.1016/0021-9797\(77\)90366-6](http://dx.doi.org/10.1016/0021-9797(77)90366-6).
- [22] J. N. Israelachvili, *Intermolecular and Surface Forces*, Academic Press, 2. edn., ISBN: 978-0-12-375182-9, 1992.
- [23] E. J. R. Parteli, J. Schmidt, C. Brümel, K.-E. Wirth, W. Peukert, T. Pöschel, Attractive particle interaction forces and packing density

- of fine glass powders, *Nature Scientific Reports* 4 (2014) 1–7, DOI: <http://dx.doi.org/10.1038/srep06227>.
- [24] X. Deng, J. V. Scicolone, R. N. Davé, Discrete element method simulation of cohesive particles mixing under magnetically assisted impaction, *Powder Technology* 243 (2013) 96–109, DOI: <http://dx.doi.org/10.1016/j.powtec.2013.03.043>.
- [25] C. Thornton, Interparticle sliding in the presence of adhesion, *Journal of Physics D: Applied Physics* 24 (1991) 1942–1946, DOI: <http://doi.org/10.1088/0022-3727/24/11/007>.
- [26] C. Thornton, K. Yin, Impact of elastic spheres with and without adhesion, *Powder Technology* 65 (1991) 153–166, DOI: [http://dx.doi.org/10.1016/0032-5910\(91\)80178-L](http://dx.doi.org/10.1016/0032-5910(91)80178-L).
- [27] M. Oda, J. Konishi, S. Nemat-Nasser, Experimental micromechanical evaluation of strength of granular materials: Effects of particle rolling, *Mechanics of Materials* 1 (1982) 269–283, DOI: [http://dx.doi.org/10.1016/0167-6636\(82\)90027-8](http://dx.doi.org/10.1016/0167-6636(82)90027-8).
- [28] K. Iwashita, M. Oda, Rolling resistance at contacts in simulation of shear band development by DEM, *Journal of Engineering Mechanics* 124 (1998) 285–292, DOI: [http://doi.org/10.1061/\(ASCE\)0733-9399\(1998\)124:3\(285\)](http://doi.org/10.1061/(ASCE)0733-9399(1998)124:3(285)).
- [29] C. Dominik, A. Tielens, Resistance to rolling in the adhesive contact of two elastic spheres, *Philosophical Magazine A* 72 (1995) 783–803, DOI: <http://dx.doi.org/10.1080/01418619508243800>.
- [30] C. Dominik, A. Tielens, The Physics of Dust Coagulation and the Structure of Dust Aggregates in Space, *The Astrophysical Journal* 480 (1997) 647–673, DOI: <http://dx.doi.org/10.1086/303996>.

- [31] S. Krijt, C. Dominik, A. Tielens, Rolling friction of adhesive microspheres, *Journal of Physics D: Applied Physics* 47 (2014) 175302 (9pp), DOI: <http://dx.doi.org/10.1088/0022-3727/47/17/175302>.
- [32] P. Gondret, E. Hallouin, M. Lance, L. Petit, Experiments on the motion of a solid sphere toward a wall: From viscous dissipation to elastohydrodynamic bouncing, *Physics of Fluids* 11 (1999) 2803–2805, DOI: <http://dx.doi.org/10.1063/1.870109>.
- [33] P. Gondret, M. Lance, L. Petit, Bouncing motion of spherical particles in fluids, *Physics of Fluids* 14 (2002) 643–652, DOI: <http://dx.doi.org/10.1063/1.1427920>.
- [34] D. Legendre, C. Daniel, P. Guiraud, Experimental study of a drop bouncing on a wall in a liquid, *Physics of Fluids* 17 (2005) 097105, DOI: <http://dx.doi.org/10.1063/1.2010527>.
- [35] C. Dritselis, Numerical study of particle deposition in a turbulent channel flow with transverse roughness elements on one wall, *International Journal of Multiphase Flow* 91 (2017) 1–18, DOI: <http://dx.doi.org/10.1016/j.ijmultiphaseflow.2017.01.004>.
- [36] V. Armenio, V. Fiorotto, The importance of the forces acting on particles in turbulent flows, *Physics of Fluids* 13 (2001) 2437, DOI: <http://dx.doi.org/10.1063/1.1385390>.
- [37] J. Marshall, *Adhesive Particle Flow - A Discrete-Element Approach*, Cambridge University Press, 1. edn., ISBN: 978-1-1-107-03207-1, 2014.
- [38] S. Benyahia, M. Syamlal, T. J. O'Brien, Extension of Hill-Koch-Ladd drag correlation over all ranges of Reynolds number and solids volume fraction, *Powder Technology* 162 (2006) 166–174, DOI: <http://dx.doi.org/10.1016/j.powtec.2005.12.014>.

- [39] J. B. McLaughlin, Aerosol particle deposition in numerically simulated channel flow, *Physics of Fluids A* 1 (1989) 1211, DOI: <http://dx.doi.org/10.1063/1.857344>.
- [40] R. Kurose, S. Komori, Drag and lift forces on a rotating sphere in a linear shear flow, *Journal of Fluid Mechanics* 384 (1999) 183–206, DOI: <http://doi.org/10.1017/S0022112099004164>.
- [41] J. B. McLaughlin, Inertial migration of a small sphere in linear shear flows, *Journal of Fluid Mechanics* 224 (1991) 261–274, DOI: <http://doi.org/10.1017/S0022112091001751>.
- [42] S. K. Robinson, Coherent motions in the turbulent boundary layer, *Annual Review of Fluid Mechanics* 23 (1991) 601–639, DOI: <http://dx.doi.org/10.1146/annurev.fl.23.010191.003125>.
- [43] J. den Toonder, F. Nieuwstadt, Reynolds number effects in a turbulent pipe flow for low to moderate Re, *Physics of Fluids* 9 (1997) 3398–3409, DOI: <http://dx.doi.org/10.1063/1.869451>.
- [44] V. Armenio, Effect of the subgrid scales on particle motion, *Physics of Fluids* 11 (1999) 3030, DOI: <http://dx.doi.org/10.1063/1.870162>.

Nomenclature

$Ad = \gamma/(\rho_p U^2 d_p)$	Adhesiveness parameter	-
$Fr = U/\sqrt{g_r d_p}$	Froude number	-
D	Pipe diameter	m
d_p	Particle diameter	m
$g_r = (1 - 1/\chi)g$	Buoyancy corrected gravity	m/s ²
$Kn = \lambda/d_p$	Knudsen number	-
L	Length of pipe section	m
p	Pressure	Pa
R	Effective particle radius	m
R	Pipe radius	m
r	Particle radius	m
r	Radial position	m
t	Time	s
$t^+ = tU/D$	Dimensionless time	-
U	Fluid bulk velocity	m/s
V	Cell volume	m ³
$Re = U\rho_f D/\mu$	Reynolds number	-
$St = \rho_p d_p^2 U/(18\mu D)$	Stokes number	-
<i>Greek letters</i>		
δ	Kroneckers delta	-
δt	Time step size	s
Δ	Cell length	m
$\Delta\gamma/\gamma$	Adhesion hysteresis parameter	-
$\epsilon = d_p/D$	Dimensionless particle diameter	-
$\lambda = E/(\rho_p U^2)$	Elasticity parameter	-
λ	Mean free molecular path	m
μ	Dynamic viscosity	kg/(m·s)
ρ	Density	kg/m ³
τ	Response time	s
$\chi = \rho_f/\rho_p$	Density ratio	-

Subscripts

f	Fluid
i,j	particle indices
mod	Stiffness-modified values
p	Particle
x, y, z	Spatial coordinates

Superscripts

–	Filtered values
+	Viscous units

Acronyms

sgs	Sub-grid scale
CFD	Computational Fluid Dynamics
DEM	Discrete Element Method
DNS	Direct Numerical Simulation
LES	Large Eddy Simulation
WALE	Wall-Adapting Local Eddy-viscosity