

# Numerical Simulation of a Magnetocaloric Heat Pump for Domestic Hot Water Production in Residential Buildings

Speaker:

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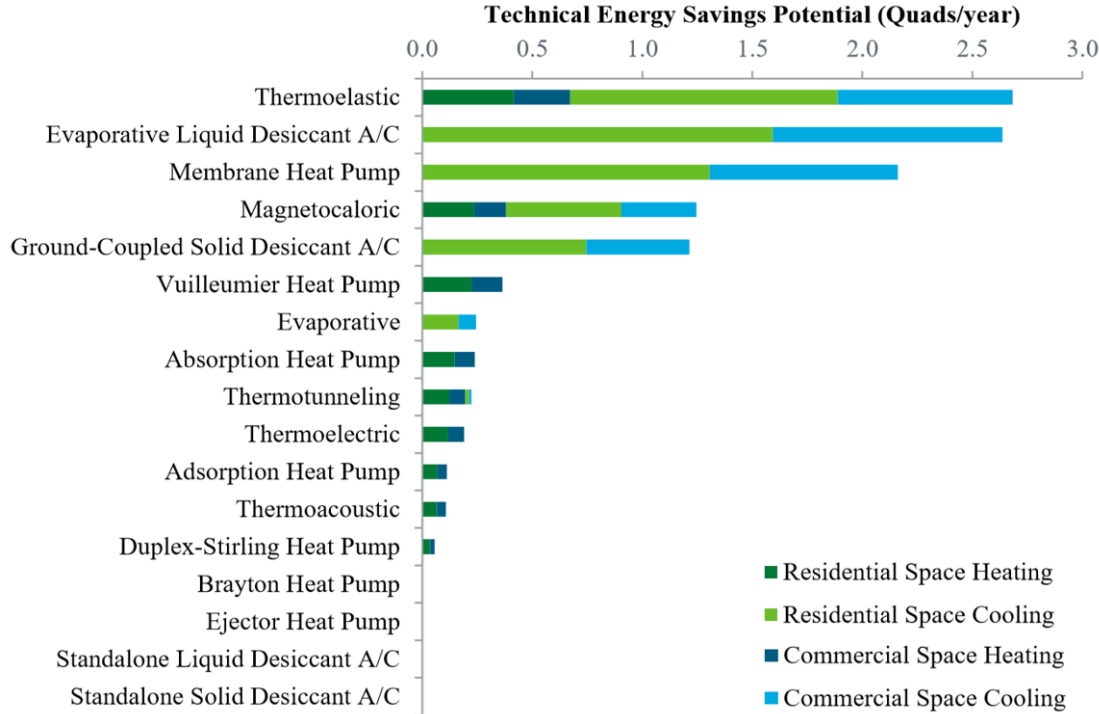
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# Motivations

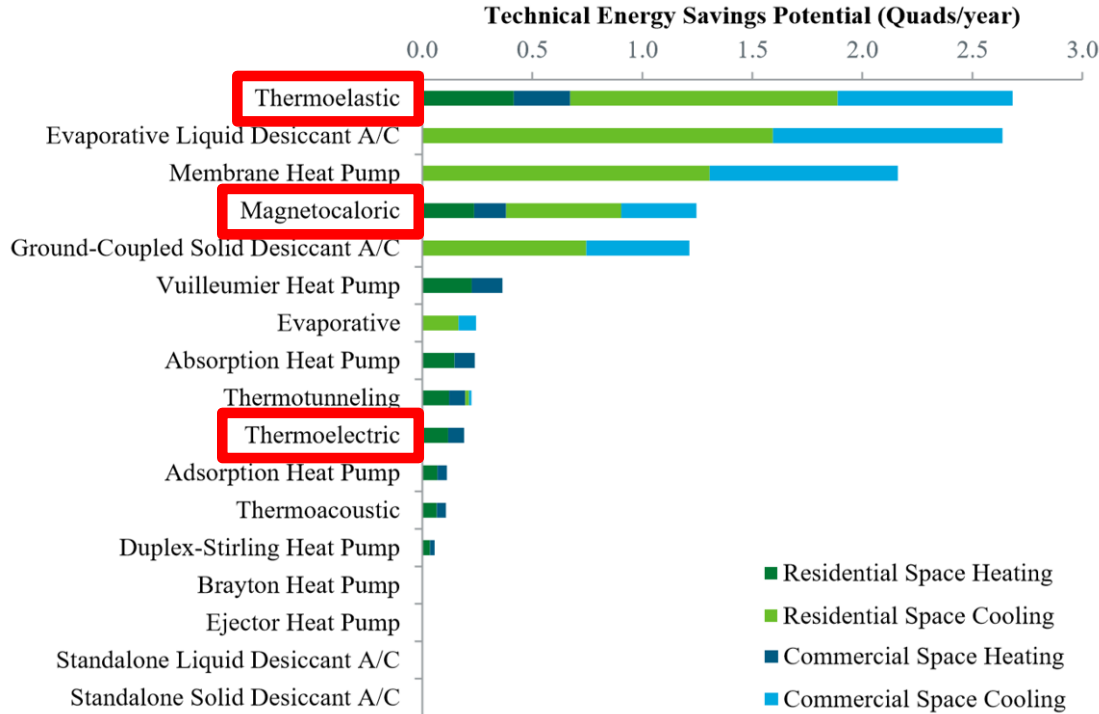
- Large increase of market demand for heat pumps
- Need to develop cost effective heating / cooling systems
- Problems with use of liquid / gas refrigerant:
  - F-gas
  - Flammability
  - Toxicity
  - Greenhouse gas effect
  - ...

# Potential of innovative systems



*Comparison of technical energy savings potential. U.S. Department of Energy, 2014.*

# Potential of innovative systems



*Comparison of technical energy savings potential. U.S. Department of Energy, 2014.*

# Caloric effects

Large adiabatic temperature change when varying a specific environment parameter of a material:

- Electrocaloric effect: variation of electrical field
- Barocaloric effect: variation of hydrostatic pressure
- Elastocaloric effect: variation of uniaxial mechanical stress
- Magnetocaloric effect: variation of magnetic field

# Potential of caloric effects

- Thermodynamic cycle to transfer heat from cold source to warmer heat sink
- Large COPs (in theory) because (nearly) reversible caloric effects
- Solid refrigerant + sustainable heat transfer fluid
- Active regenerator cycle to achieve temperature span above adiabatic temperature change of caloric effects

# Challenges

- Not mature technologies compared to vapor-compression
- Has to prove its competitiveness

# Objectives of the ENOVHEAT project

Create a magnetocaloric heat pump prototype for a single-family house in Denmark:

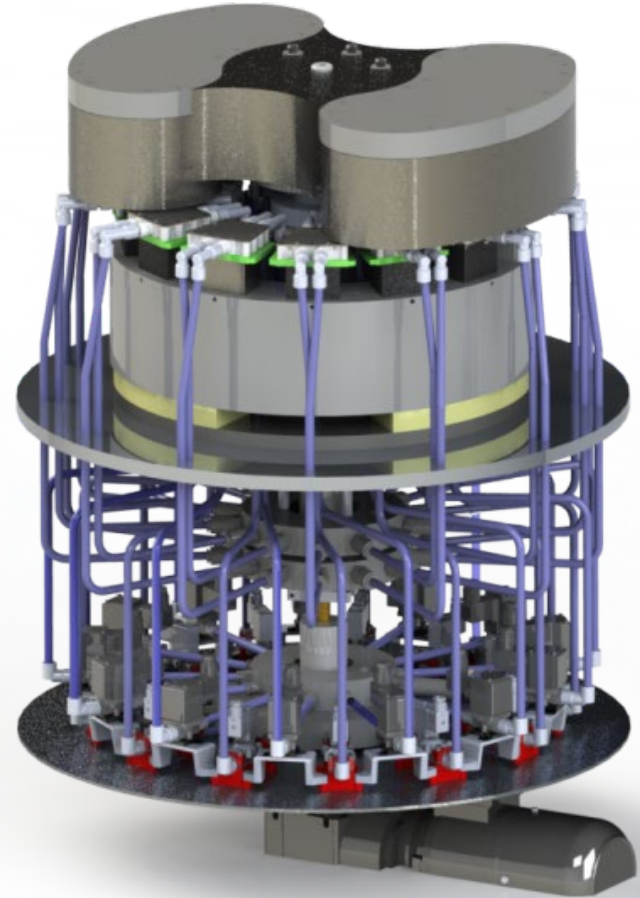
- Provide indoor space heating
- 1 - 1.5 kW of heating power
- 20 - 25 K of temperature span
- COP of 5





# Magnetocaloric heat pump

The ENOVHEAT  
magnetocaloric heat pump  
prototype:  
*"MagQueen"*



# Magnetocaloric effect

- The most studied and developed of all caloric effects
- Reversible temperature change in a magnetocaloric material magnetized or demagnetized:
  - Warms up when magnetic field is applied
  - Cools down when magnetic field is removed
- Thermodynamic cycle to transfer heat from cold source to warmer heat sink: active magnetic regenerator cycle

# Magnetocaloric heat pump

Packed sphere bed of magnetocaloric material:

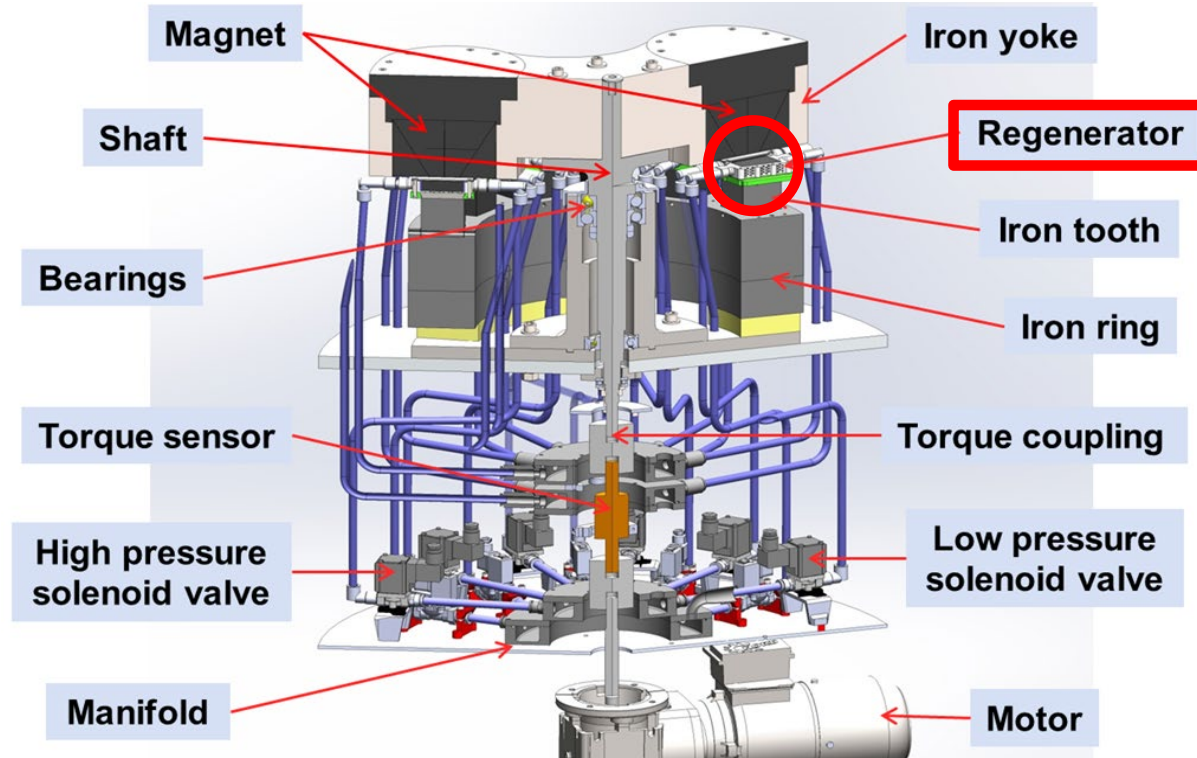
- Gadolinium
- $\text{La}(\text{Fe}, \text{Mn}, \text{Si})_{13}\text{H}_y$

Hot side

Cold side

Regenerator casing:  
Heat exchanger

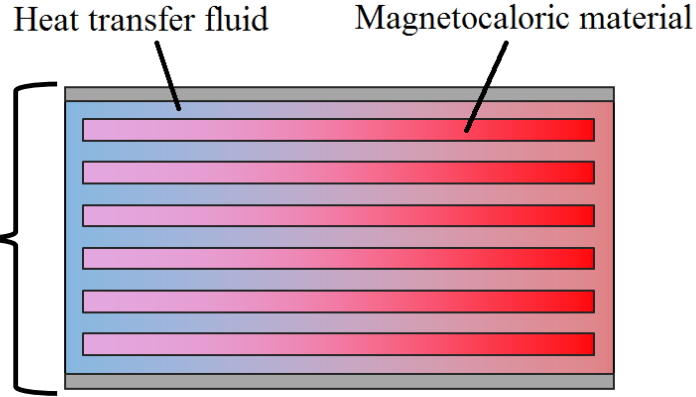
# Magnetocaloric heat pump



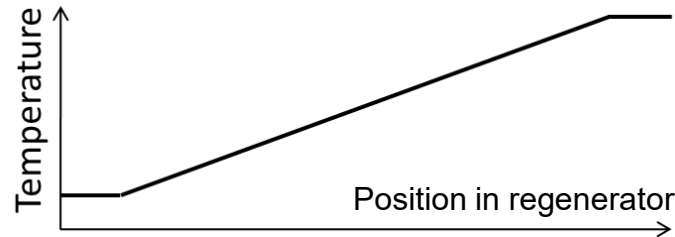
# Active magnetic regenerator cycle

**Cold side:  
Heat source**

Regenerator

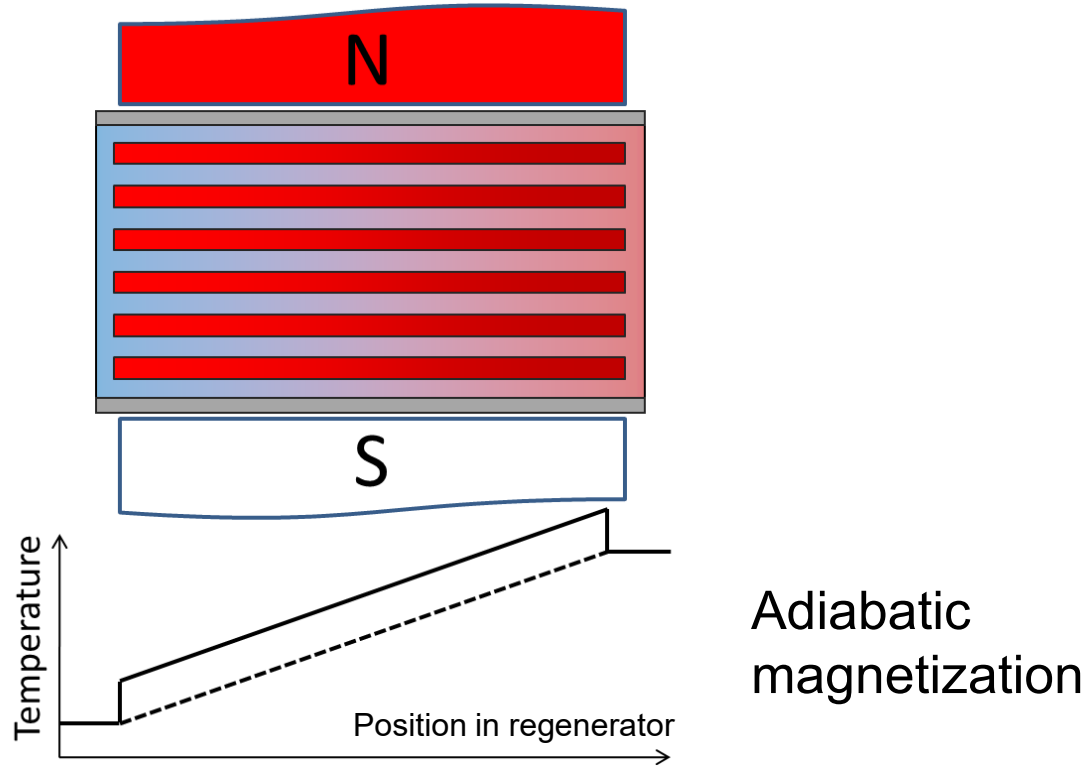


**Hot side:  
Heat sink**

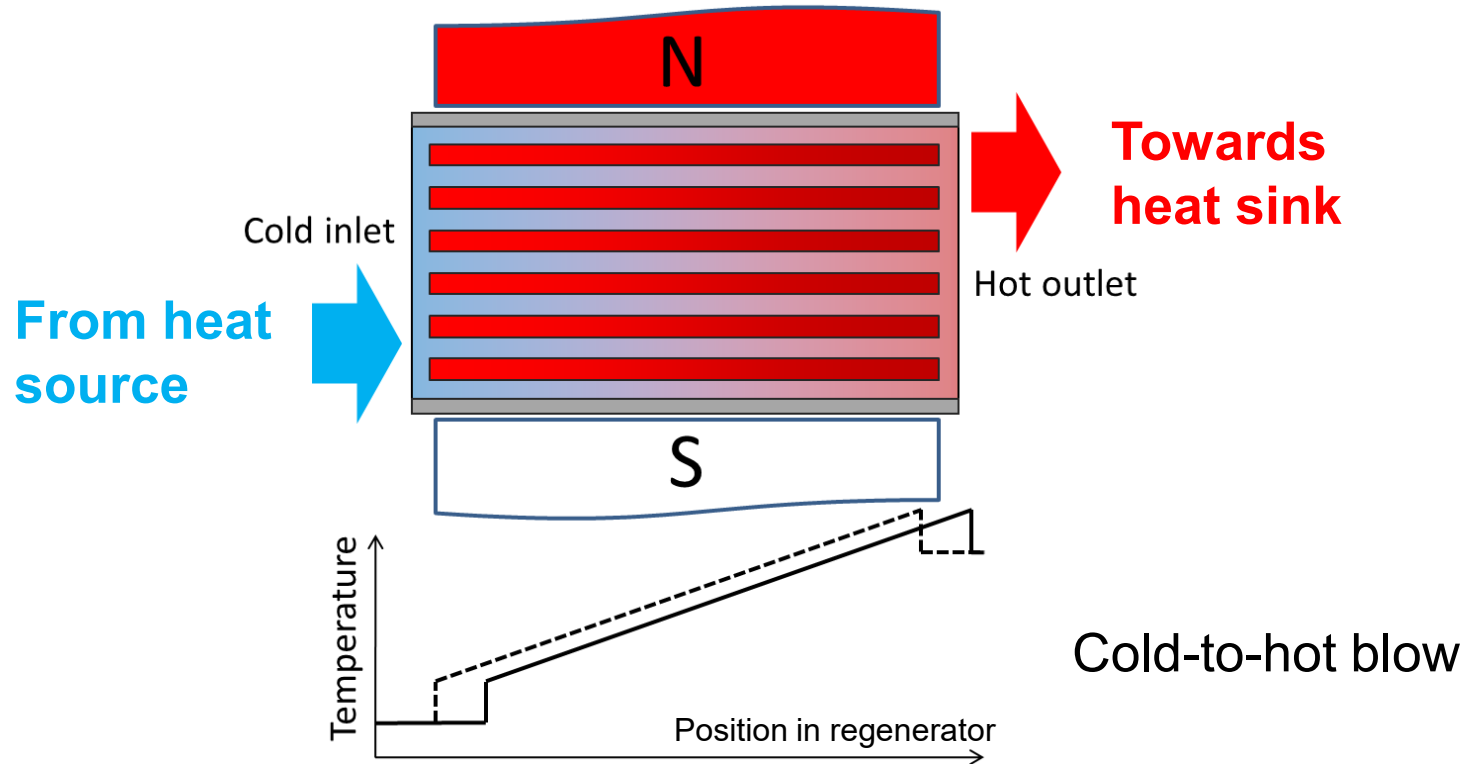


Initial state with  
temperature gradient

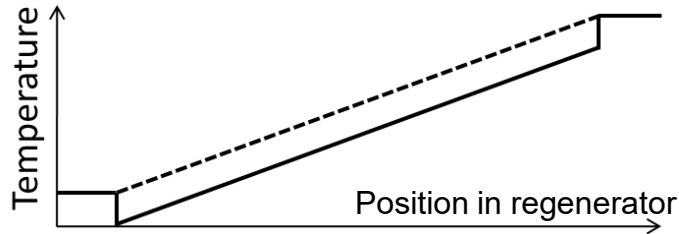
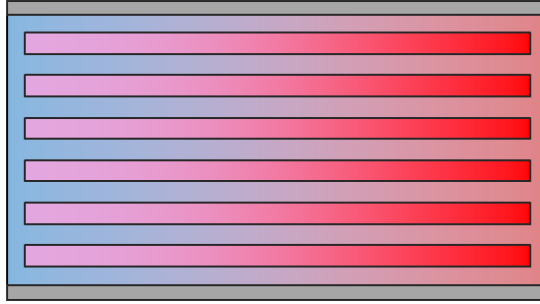
# Active magnetic regenerator cycle



# Active magnetic regenerator cycle



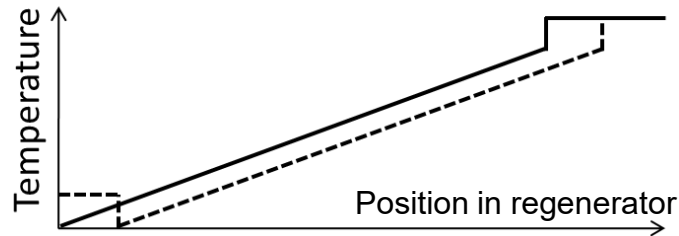
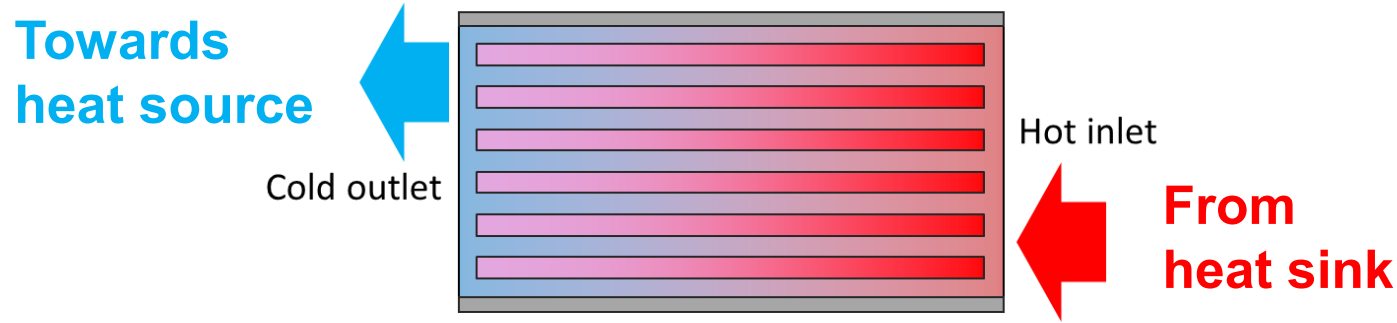
# Active magnetic regenerator cycle



Adiabatic  
demagnetization

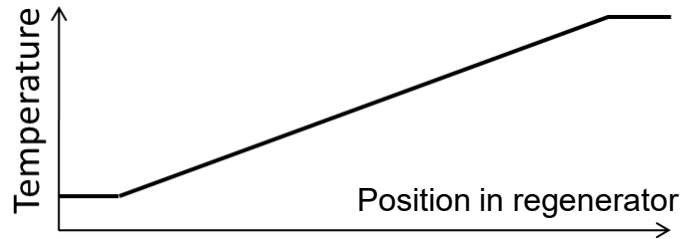
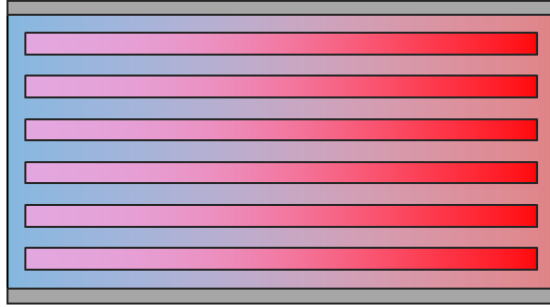


# Active magnetic regenerator cycle



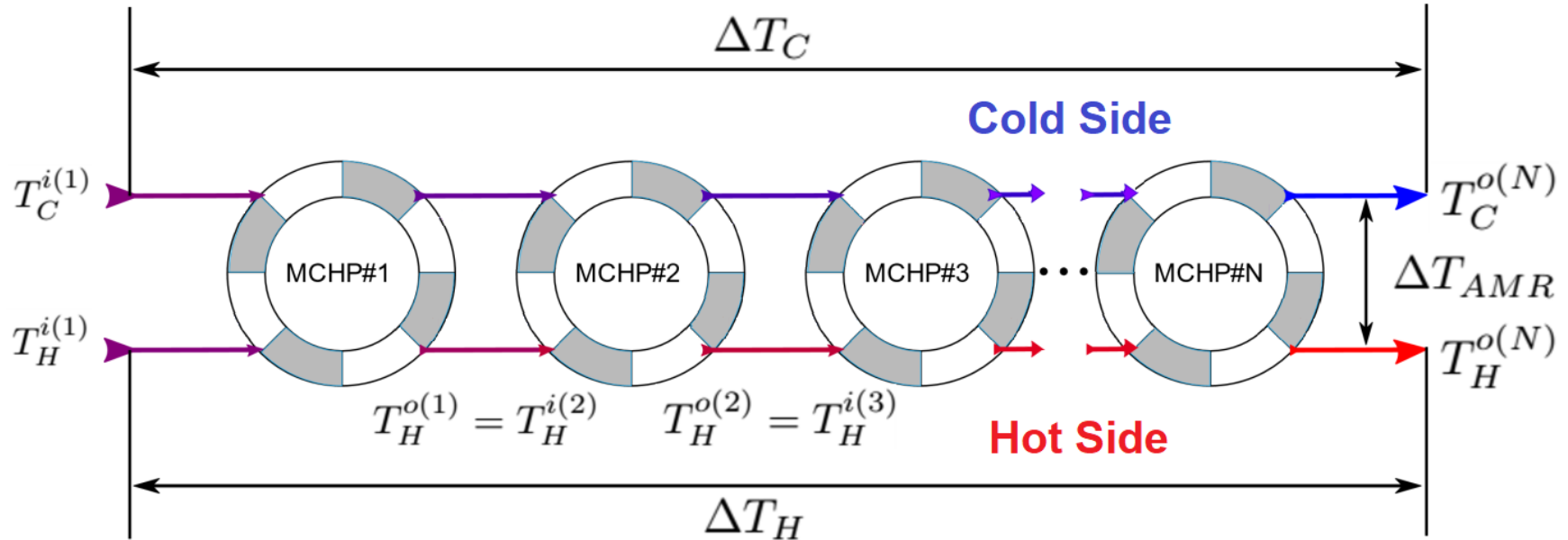
Hot-to-cold blow:  
heat regeneration

# Active magnetic regenerator cycle



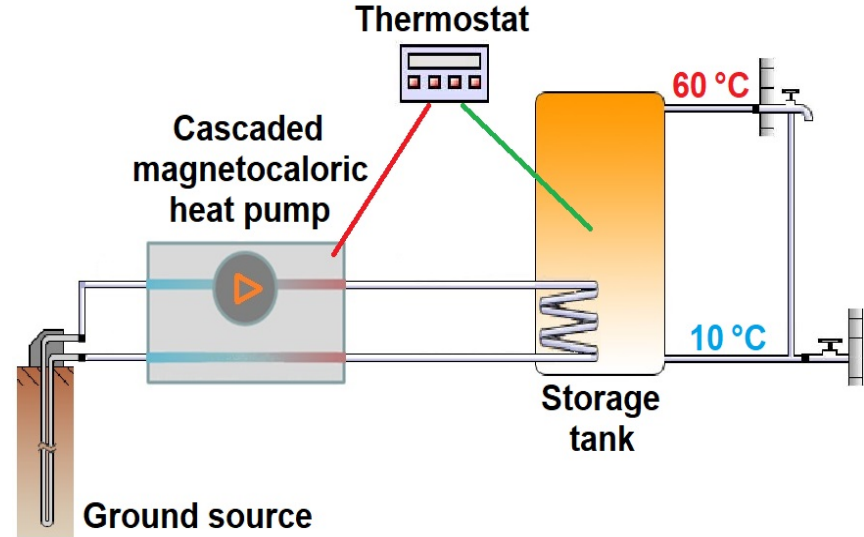
Back to initial state

# Cascading regenerators: increase temperature span



# Objectives of this numerical study

Test different cascading configurations for higher temperature span for Domestic Hot Water production in a Danish single-family house.



## Study case

- Hot water draw-off profile from a single-family house in Denmark: 60 °C
- 250 L hot water storage tank
- 5 cascaded heat pump configurations:
  - 2 – 24 cascaded heat pumps
  - 1 – 12 regenerators each
- Magnetocaloric material:  $\text{La}(\text{Fe},\text{Mn},\text{Si})_{13}\text{H}_y$
- Vertical borehole ground source

# Numerical modelling

$$\frac{\partial}{\partial x} \left( k_{\text{disp}} A_c \frac{\partial T_f}{\partial x} \right) - \dot{m}_f c_f \frac{\partial T_f}{\partial x} - \frac{Nuk_f}{d_h} a_s A_c (T_f - T_s) + \left| \frac{\partial P}{\partial x} \frac{\dot{m}_f}{\rho_f} \right| = A_c \varepsilon \rho_f c_f \frac{\partial T_f}{\partial t}$$

$$\frac{\partial}{\partial x} \left( k_{\text{stat}} A_c \frac{\partial T_s}{\partial x} \right) + \frac{Nuk_f}{d_h} a_s A_c (T_f - T_s) = A_c (1 - \varepsilon) \rho_s \times \left[ c_H \frac{\partial T_s}{\partial t} + T_s \left( \frac{\partial s_s}{\partial H} \right)_{T_s} \frac{\partial H}{\partial t} \right]$$

*Detailed heat pump model  
[Engelbrecht and Lei]*

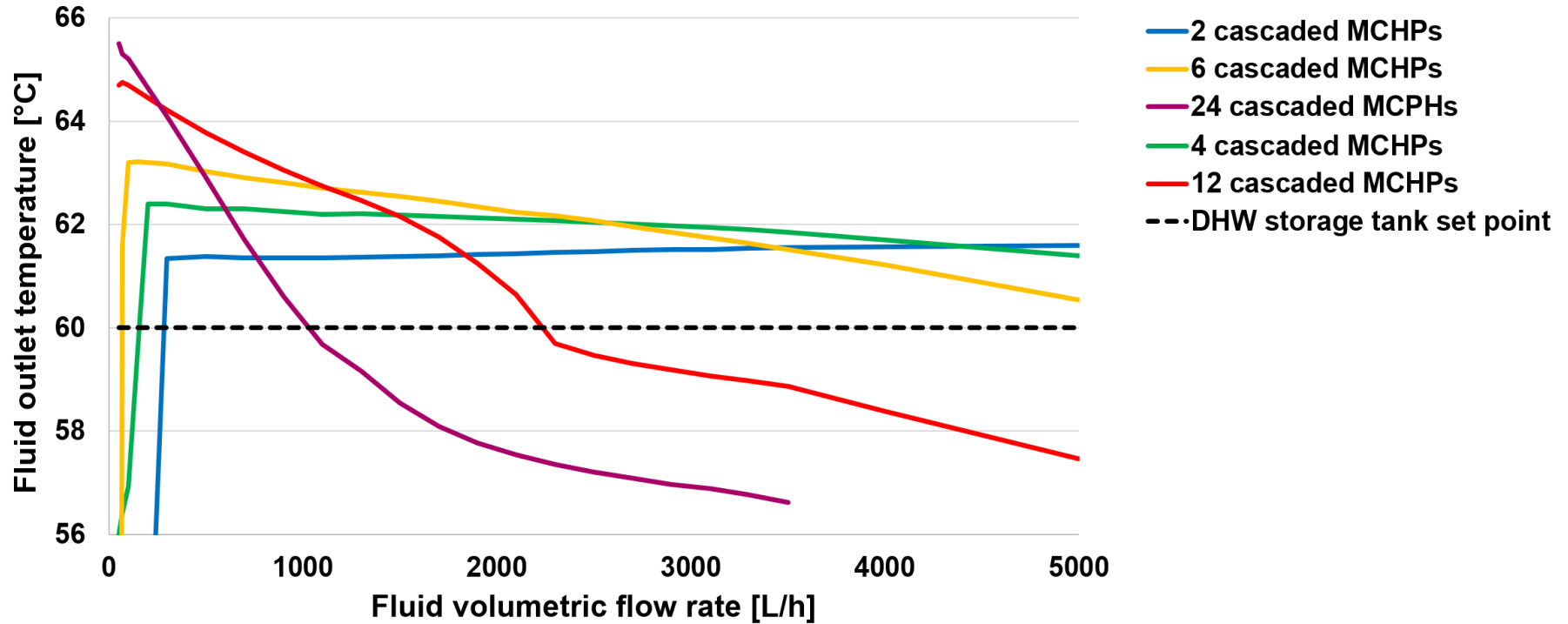


5-dimensional  
lookup tables  
(1600 points)

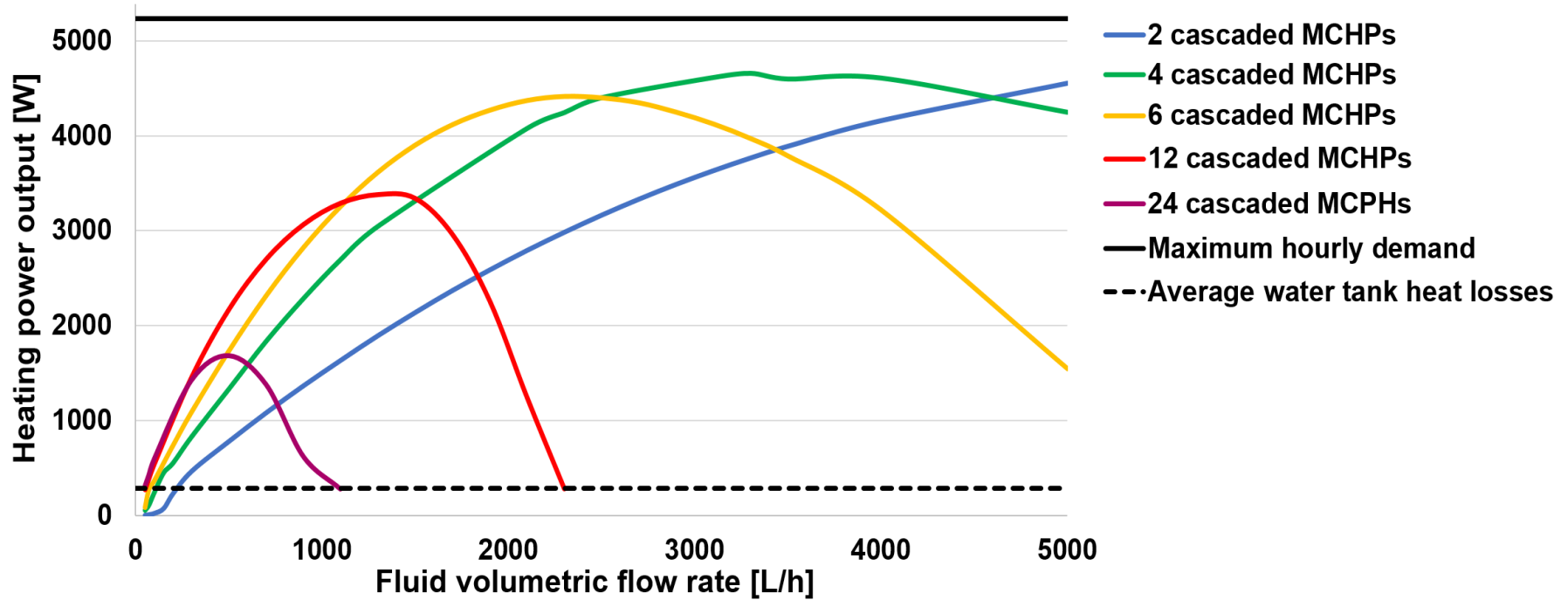


MATLAB-Simulink  
model of ground source,  
storage tank and piping

# Results: outlet fluid temperature

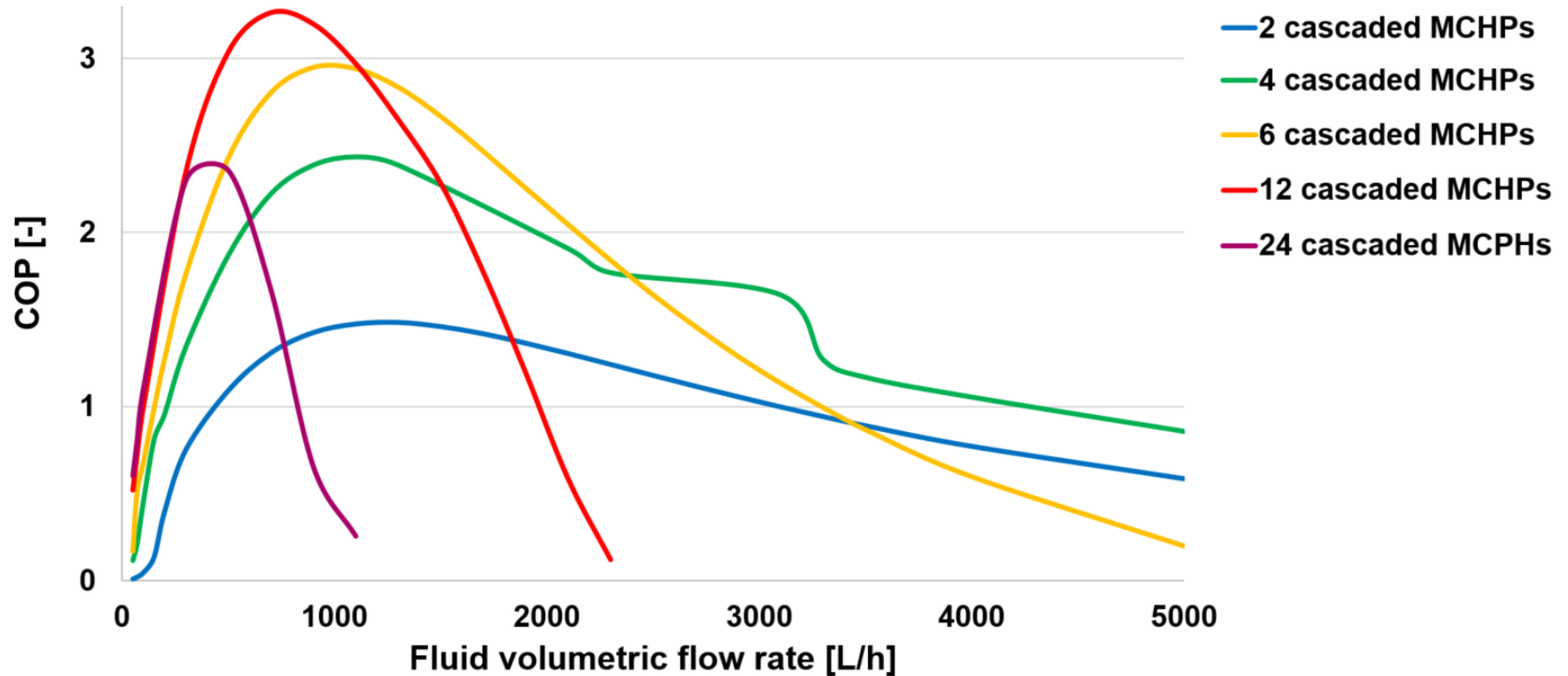


# Results: useful heating power output

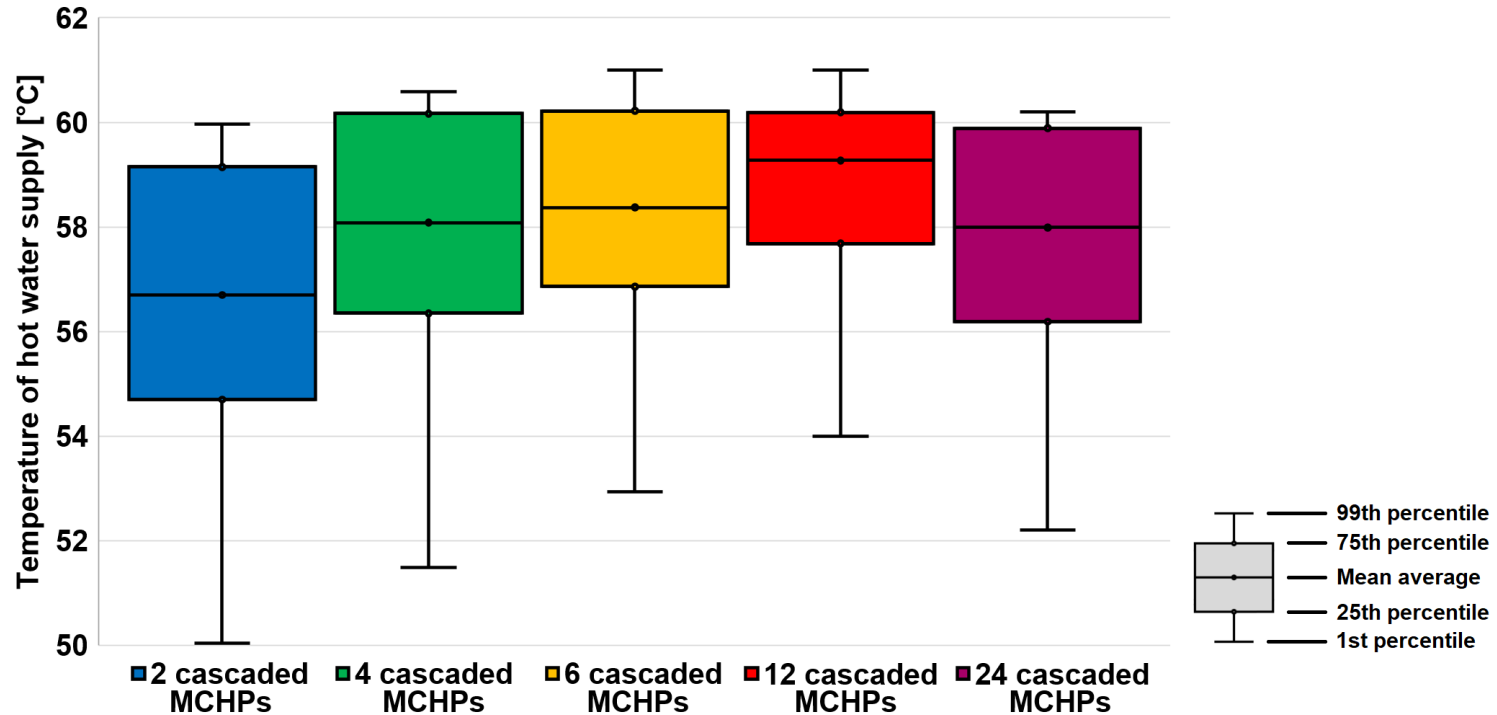




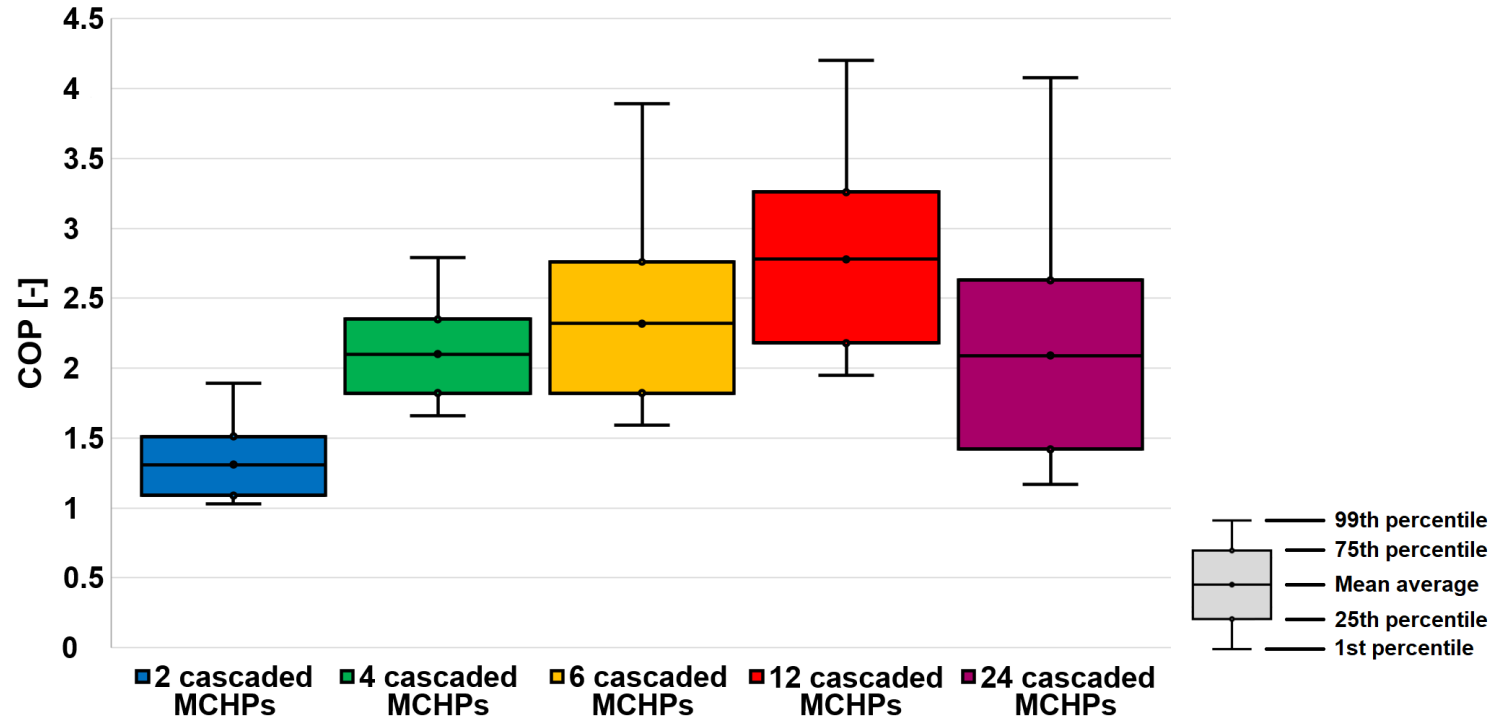
# Results: system COP



# One-month test results: hot water draw-off temperature



# One-month test results: COP



# Conclusion

- A cascaded magnetocaloric heat pump system can produce a sufficient temperature span for the production of domestic hot water at 55 °C – 60 °C
- Possible to produce hot water need for a single-family house
- Monthly average COP up to 2.78

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## Questions and Comments

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